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1	Performance and Emission Analysis of Diesel Engine using CNG
2	under Dual Fuel Mode with Exhaust Gas Recirculation
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#### 7 Abstract

8 An experimental investigation was carried out to find out the performance and emissions of a

<sup>9</sup> diesel engine operated with CNG inducted into the engine and compared with that of using

<sup>10</sup> exhaust gas recirculation. A single cylinder, 4 stroke, and compression ignition engine was

<sup>11</sup> used. Behavior of the engine at 10

12

13 Index terms— Compressed Natural Gas, Emissions, UHC.

### 14 1 Introduction

ompressed Natural Gas (CNG) has become a better option as a clean burning fuel of an IC engine. In order 15 to comply with the ever-stringent emission norms throughout the world and crunch in petroleum reserves, the 16 modern day automobile industry is compelled to hunt for new and alternative means of fuel sources to keep the 17 wheels spinning globally [1]. Paradoxical objectives of attaining simultaneous reduction in emissions along with 18 high performance has provided with a few alternative. Natural gas produces practically no particulates since it 19 contains few dissolved impurities (e.g. sulphur compounds). Moreover, natural gas can be used in compression 20 ignition engines (dual fuel diesel-natural gas engines) since the auto-ignition temperature of the gaseous fuel is 21 higher compared to the one of conventional liquid diesel fuel [3]. 22

Dual fuel diesel-natural gas engines feature essentially a homogeneous natural gas-air mixture compressed 23 rapidly below its auto-ignition conditions and ignited by the injection of an amount of liquid diesel fuel around 24 top dead center position. Natural gas is fumigated into the intake air and premixed with it during the induction 25 stroke. At constant engine speed, the fumigated gaseous fuel replaces an equal amount of the inducted combustion 26 air (on a volume basis) since the total amount of the inducted mixture has to be kept constant. Furthermore, 27 under fumigated dual fuel operating mode, the desired engine power output (i.e. brake mean effective pressure) is 28 controlled by changing the amounts of the fuels used. Thus, at a given combination of engine speed and load, the 29 change of the liquid fuel "supplementary ratio" leads to a change of the inhaled combustion air, thus resulting to 30 the alteration of the total relative air-fuel ratio [1][2][3]. In internal combustion engines, exhaust gas recirculation 31 (EGR) is a nitrogen oxide (NOx) emissions reduction technique used in petrol/gasoline and diesel engines. EGR 32 works by re-circulating a portion of an engine's exhaust gas back to the engine cylinders [5]. In a gasoline engine, 33 this inert exhaust displaces the amount of combustible matter in the cylinder. In a diesel engine, the exhaust gas 34 replaces some of the excess oxygen in the pre-combustion mixture. Because NOx forms primarily when a mixture 35 of nitrogen and oxygen is subjected to high temperature, the lower combustion chamber temperatures caused 36 by EGR reduces the amount of NOx the combustion generates. Most modern engines now require exhaust gas 37 recirculation to meet the emission standard [4, [6]][7][8][9]. 38

## <sup>39</sup> 2 Experimental Procedure

<sup>40</sup> Series of several experimental cycles have been conducted with varying CNG percentages and iterations were <sup>41</sup> done with varying exhaust gas recirculation and the results were compared. The engine used in the present study

<sup>42</sup> is a Kirloskar AV-1, single cylinder direct injection, Water cooled diesel engine with the specifications given in <sup>43</sup> Table ??0 1. Diesel injected with a nozzle hole of size 0.15mm.the engine is coupled to a dynamometer. Engine exhaust emission is measured. Load was varied from 0.5 kilo watt to 3 kilo watts. The amount of exhaust gas
sent to the inlet of the engine is varied. At each cycle, the engine was operated at varying load and the efficiency
of the engine has been calculated simultaneously.

The experiment is carried out by keeping the compression ratio constant i.e., 16.09:1. The exhaust gas 47 analyzer used is MN-05 multi gas analyzer shown in Fig. 1. (4 gas version) is based on infrared spectroscopy 48 technology with signal inputs from an electrochemical c cell. Non-dispersive infrared measurement techniques use 49 for CO, CO2, and HC gases. Each individual gas absorbs infrared radiation absorbed can be used to calculate 50 the concentration of sample gas. Analyzer uses an electrochemical cell to measure oxygen concentration. It 51 consists of two electrodes separated by an electrically conducted liquid or cell. The cell is mounted behind a 52 polytetrafluorethene membrane through which oxygen can diffuse. The Device therefore measures oxygen partial 53 pressure. If a polarizing voltage is applied between the electrodes the resultant current is proportional to the 54 oxygen partial pressure. The important properties of diesel fuel and natural gas are given in Table ??. 55 Table ?? : Properties of Diesel Fuel and Natural Gas a Natural gas consists of various gas species; from which 56

57 methane (CH4) is the main constituent. The equivalent chemical composition of natural gas may be expressed 58 as C1.16H4.32 [10]. b At normal temperature and pressure. From the chart1 it can be seen that up to 40% CNG 59 substitution would be observed an increase in brake thermal efficiency of 10% compared to that of pure diesel, 50% substitution of CNG has shown 5% decrease in brake thermal efficiency when compared to that of pure 51 diesel.

# <sup>62</sup> 3 Engine Specifications

63 Chart 2 : Bth Vs EGR% at various CNG proportions The chart 2 represents the relation between brake thermal 64 efficiency and percentage of Exhaust gas recirculation. it shows that with an increase in exhaust gas recirculation 65 proportion the brake thermal efficiency has increased till 10% of substitution but decreased with above 10% 66 substitution . The relation between volumetric efficiency and exhaust gas recirculation is represented in chart 67 3. It has been observed that the volumetric efficiency decreases with an increased substitution of compressed 68 natural gas (CNG) and with increased exhaust gas recirculation (EGR)

<sup>68</sup> natural gas (CNG) and with increased exhaust gas recirculation (EGR).

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## <sup>70</sup> 5 Chart 4 : NOx Vs EGR% at various proportions

The chart 4 represented the trends of NOx with the EGR substitution. it is observed that, with an increase in exhaust gas recirculation NOx emission havedecreased by 28% at all proportions of CNG substitution. Chart 8 : in-cylinder pressure Vs crank angle for pure diesel, 10%CNG and 10%EGR substitution Chart 8 shows the pressure inside cylinder at varying crank angles of the cycle for pure diesel, 10% CNG substitution and 10% exhaust gas recirculation at 10% CNG substitution.

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## 78 8 Conclusion

From the above obtained results the following conclusions were drawn: a) Substitution of CNG up to 40% has shown increase on brake thermal efficiency of 20% compared to that of pure diesel, but 50%<sup>1</sup>

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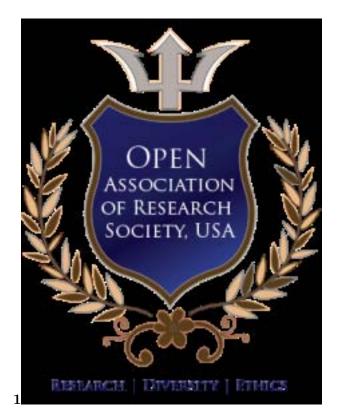
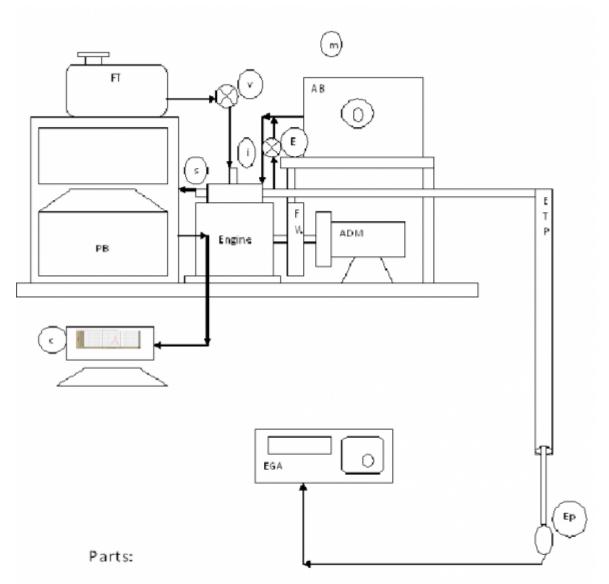


Figure 1: Figure 1 :



Figure 2: Figure 2 :



AB-air box ,E-Exhaust Gas recirculation perocision,mmeasurement of air by mano meter , Fw-fly wheel, ADMalternator dynamometer, i-fuel injector,C-computer for P-0

Figure 3:

 $\begin{array}{c} \text{Year} \\ 2014 \end{array}$ 

32  $\qquad$  Fuel Chemical formula Density (kg/m 3 ) Low heating value (MJ/kg) Flammability limits (% vol.)

	Cetane number	52  N/A
	Autoignition temperature	220580
( ) A	(0  C) (AFR stoic , kg air/kg	14.36.82
Vol	fuel) Stoichiometric air-fuel ra-	
	tio	
Global		
Jour-		
nal of		
Re-		
searches	3	
in		
Engi-		
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ing		

[Note:  $\bigcirc$  2014 Global Journals Inc. (US)]

Figure 4: Table 2 :

#### 8 CONCLUSION

#### 3

		Year $2014$
		33
		() A Vol
NO X B th Vol. Eff. UHC PPM EGR	Oxides of nitrogen Brake ther-	Global
	mal efficiency Volumetric Effi-	Journal of
	ciency Unburnt hydro carbons	Researches
	Parts per million Exhaust Gas	in Engi-
	Recirculation	neering
CA	Crank Angle	
C. Brake Thermal Efficiency		
Chart1 represents the trends of brake		
the rmal		
efficiency with the substitution of		
compressed natural		
gas (CNG) with corresponds to Brake		
power		
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Figure 5: Table 3 :

shown 11% decrease in cylinder pressure (bar) CA (Deg) in cylider pressure Vs crank angle for pure diesel, 10%CNG and 10%EGR substitutoin in brake thermal efficiency when compared b) to that of diesel. The normal injection timing has shown higher volumetric efficiency. Any how the trend of varying volumetric efficiency has stood very general. substitution of CNG has

Figure 6:

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