

Performance Improvement of a Domestic Refrigerator by using PCM (Phase Change Material)

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Abstract

The paper investigates the performance improvement provided by a phase change material associated with the evaporator in a domestic refrigerator. The heat release and storage rate of a refrigerator is depends upon the characteristics of refrigerant and its properties. The usage of PCM as TS will help to improve the COP (Coefficient of performance) of new refrigeration cycle by introducing a new sub cooling routine. The analysis of the experiment exemplifies the improvement of the system coefficient of performance considerably. Using water as PCM and for a certain thermal load it is found that the coefficient of performance of the conventional refrigerator increased by 55-60

Index terms— phase change material, refrigerator, cop, compressor.

1 Introduction

he most alarming environmental disorder namely "Global Warming" refers to the rising temperature of Earth's atmosphere and ocean and its projected continuation. The heat from the Sun is entrapped in the Earth and thus increases the temperature of the atmosphere by Green house Effect. Refrigeration system is directly and invisibly responsible for Global Warming problem. For the typical home of the early 1990s, a frost-free refrigerator or freezer was the second most expensive home appliance to operate besides the water heater. Appliance makers were required to include labels listing an estimate of the annual cost of running each appliance so consumers could compare costs and energy usage. [1] A refrigerator (colloquially fridge) is a common household appliance that consists of a thermally insulated compartment and a heat pump (mechanical, electronic, or chemical) that transfers heat from the inside of the fridge to its external environment so that the inside of the fridge is cooled to a temperature below the ambient temperature of the room. [2] Domestic refrigerators are among the most energy-demanding appliances in a household due to their continuous operation. [3] The domestic refrigerator is one found in almost all the homes for storing food, vegetables, fruits, beverages, and much more. [4] Materials that can store thermal energy reversibly over a long time period are often referred to as latent heat storage materials.

2 III. Overview of Phase Change Material (PCM)

PCMs latent heat storage can be achieved through solid-solid, solid-liquid, solid-gas and liquid-gas phase change. However, the only phase change used for PCMs is the solid-liquid change.

Thermal Energy Storage through Phase Change material has been used for wide applications in the field of air conditioning and refrigeration especially at industrial scale. [6] A phase-change material (PCM) is a substance with a high heat of fusion which, melting and solidifying at a certain temperature, is capable of storing and releasing large amounts of energy. [7] Even though the thermal conductivity of phase change materials (PCM) is usually not high, it is sufficient to enhance the global heat transfer conditions of an evaporator with air as external fluid and natural convection as heat transfer mechanism. [8] The vapor-compression uses a circulating liquid refrigerant as the medium which absorbs and removes heat from the space to be cooled and subsequently rejects that heat elsewhere. Figure depicts a typical, single-stage vapor compression system. All such systems

have four components: a compressor, a condenser, a Thermal expansion valve (also called a throttle valve or Tx Valve), and an evaporator. Circulating refrigerant enters the compressor in the thermodynamic state known as saturated vapor and is compressed to a higher pressure, resulting in a higher temperature as well. [9] Figure 1: Vapor compression refrigeration system V.

3 USING PCM AS LATENT HEAT STORAGE SYSTEM

In the conventional household refrigerator the compressor works in ON/OFF mode. The refrigerant of the evaporator coil takes the cabinet heat during compressor ON mode. If PCM is used in the cabinet then it will take most of the heat by changing its phase from solid to liquid. The temperature is constant until the melting process is finished. Moreover, if the PCM is touched with the evaporator coil the stored heat energy of PCM will be extracted by the refrigerant through conduction method during compressor on mode. The conduction transfer is faster than the natural convection heat transfer. In the conventional refrigerator the cabinet heat is extracted by the refrigerant through natural convection. So the PCM will improve the heat transfer performance of the evaporator also.

A mathematical model of parallel plate's field with a phase change material that absorbs heat from the flow of warm moist air was developed and validated. In this study, effects of the design and the operating condition on the performance of the system are discussed only for the melting process and the interaction with the refrigeration system is not studied. ??

4 Data Collection and Result

The following data have been collected for each test run at the steady state condition of the system.

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5 Discussion

Experiments were carried out under certain thermal loads with water as PCM. Here the effect of PCM in certain quantities in this case 5 liters at certain thermal loads on the performance parameter of house hold refrigerator. The number of compressor on-off cycle within a certain period of time for different PCMs and without PCM can be pointed up. Use of water as PCM imposes a great impact on COP improvement at certain thermal loads. Using water as PCM and certain thermal load it is found that the 55-60% COP improvement has been achieved by the PCM in respect without PCM in conventional refrigerator.

During the compressor running the refrigerant takes the chamber heat by free convection in case of without PCM, which is slower heat transfer process in respect to conduction process. But PCM most of the heat in the cabinet is stored in the PCM during compressor running time. Since the conduction heat transfer process is faster than the free convection process the cooling coil temperature does not require dropping very low to maintain desired cabinet temperature. As a result the evaporator works at high temperature and pressure with PCM. Moreover, due to high operating pressure and temperature of the evaporator the density of the refrigerant vapor increases, as a result the heat extracted from the evaporator by the fixed volumetric rate compressor is higher than without PCM.

6 IX.

7 Conclusion

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Figure 1:

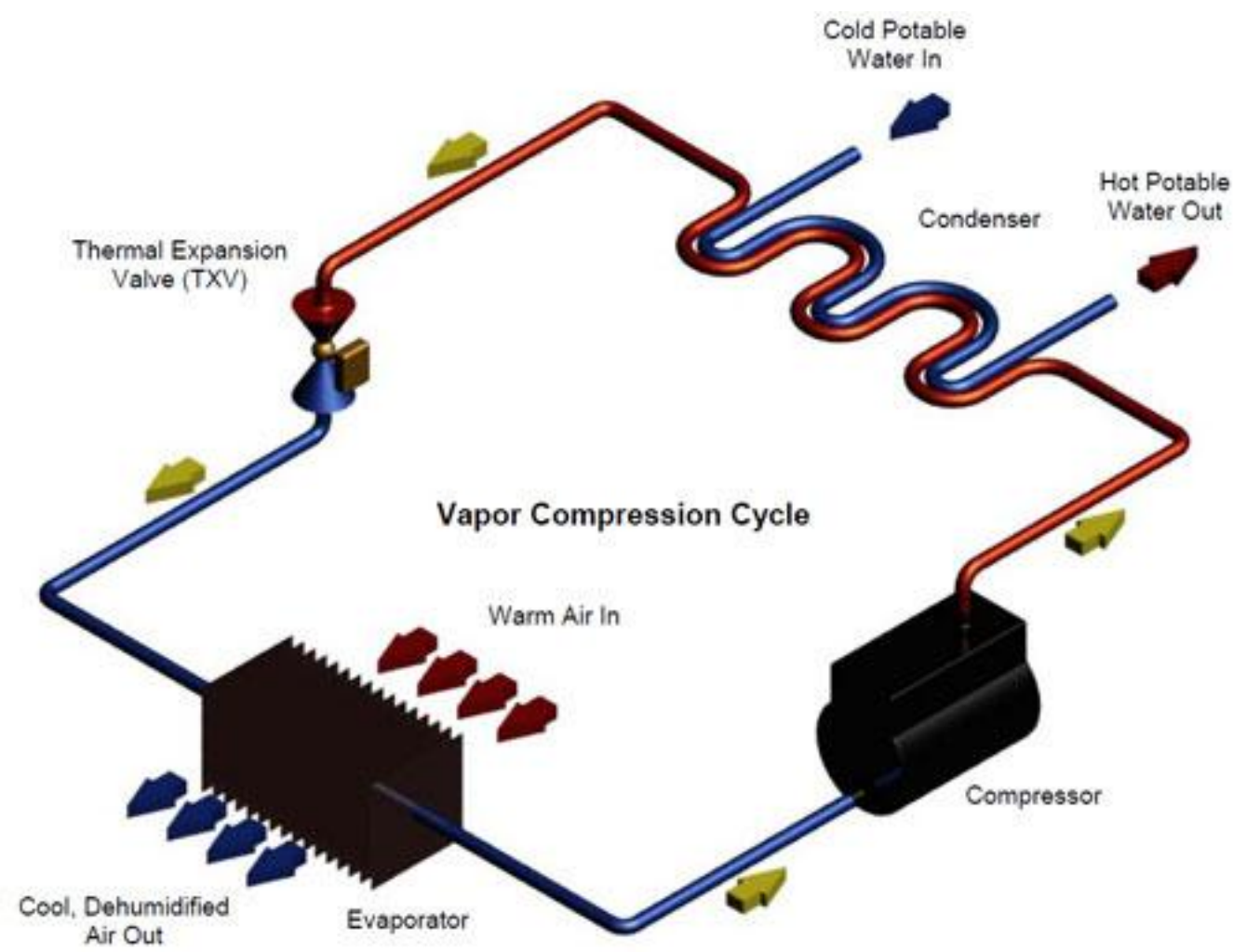
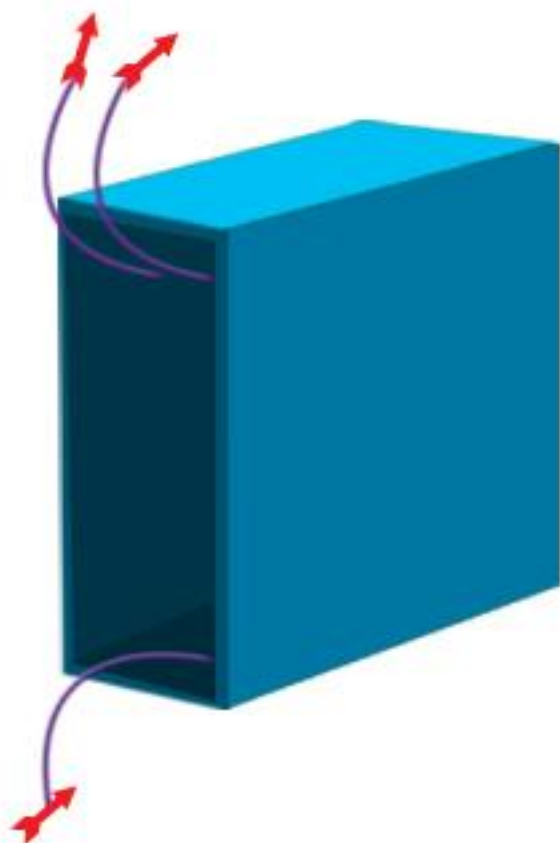


Figure 2:



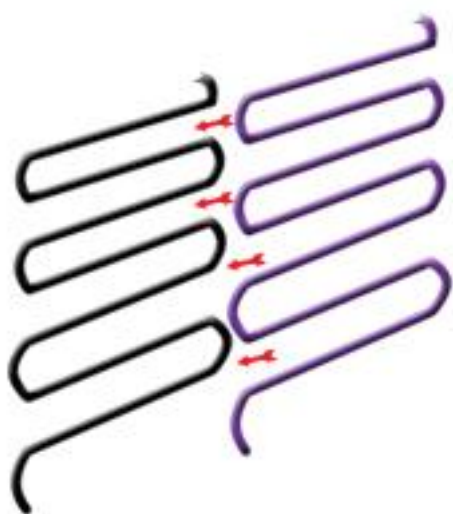
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Figure 3: Figure 2 :Figure 3 :



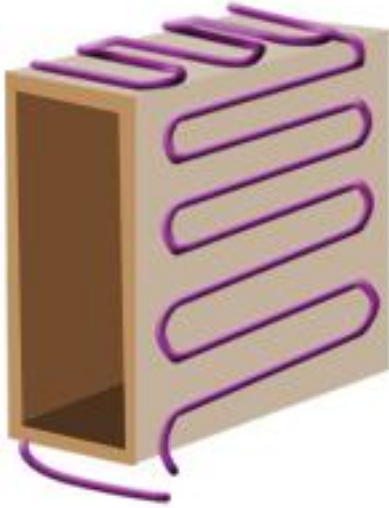
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Figure 4: Figure 4 :Figure 6 :



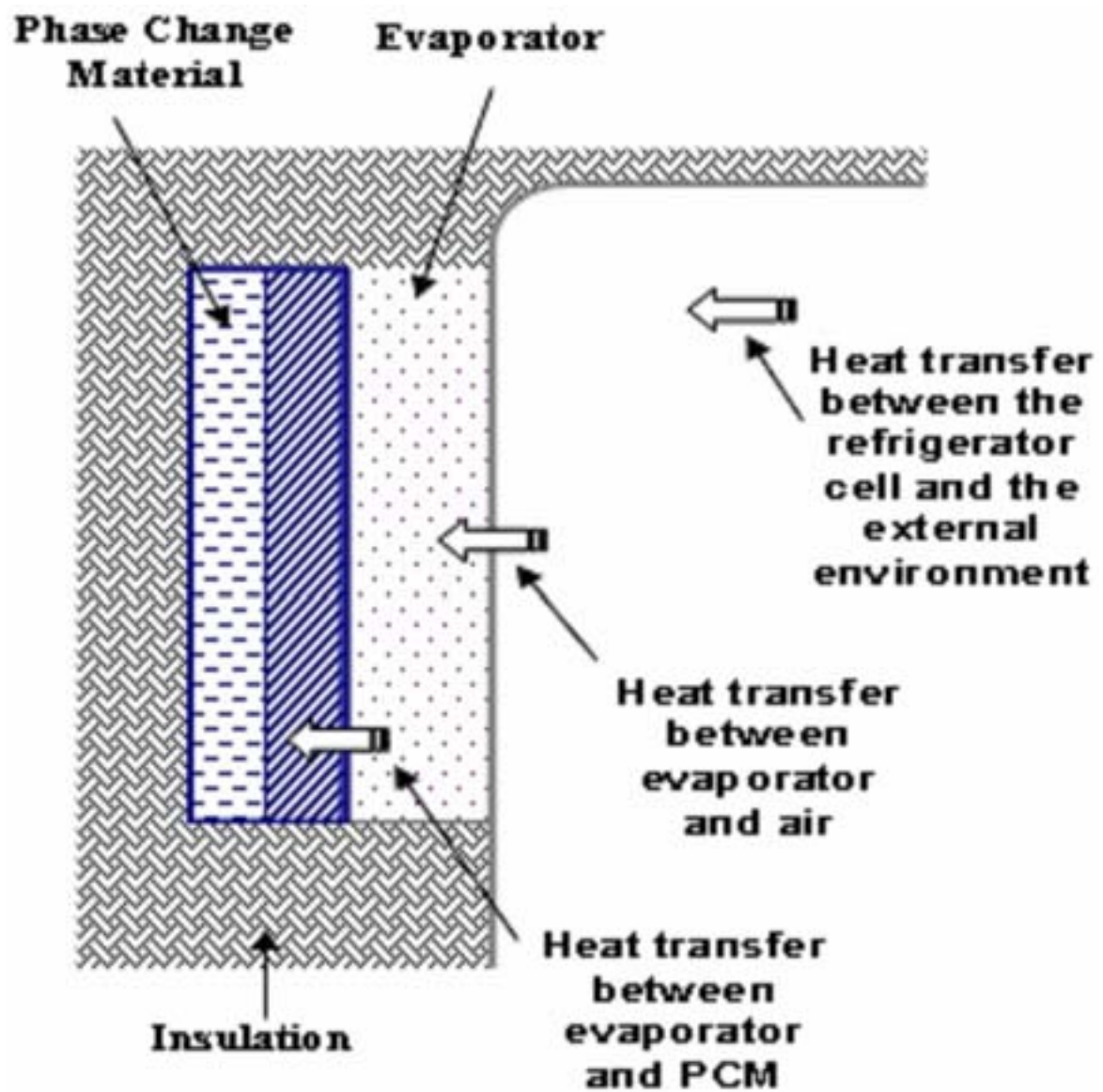
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Figure 5: P 1 =



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Figure 6: Figure 5 :



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Figure 7: ?? At step 6 COP

$$\frac{h_1 - h_4}{h_2 - h_1} \quad [9]$$

Figure 8:

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Figure 9: Table 1 :

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	Time Reading		Evaporator		Condenser		Compressor		Compressor		Condenser		Condenser	
	taken		inlet		outlet		inlet		outlet		inlet		outlet	
			Pressure		Pressure		Temp		Temp		Temp		Temp	
			P1 bar		P3 bar		T1 ? C		T2 ? C		T3? C		T4? C	
	11.10													
	am		0.27		9		26		50		33		17	
	11.25													
	am		0.4		8.5		27		53		36		17	
013 2	11.40 am		0.44		9		26		58		37		19	
Year	11.55 am		0.47		9.5		27		61		37		18	
20	12.1													
	pm		0.51		10		27		65		39		21	
Volume	12.25 pm	12.40	0.34	0.44	10.2	10.2	26 22 22		67 72 72		38 41 39		19 18 18	
XIII	pm	12.55pm	0.44	0.57	10.2	10.4	20 20		72 72		41 42		19 19	
Issue	1.10 pm	1.25	0.57		10.6									
X Ver-	pm													
sion														
I														
()	Time	Reading	Evaporator		Condenser		Compressor		Compressor		Condenser		Condenser	
A Re-	taken	10.00 am	inlet		outlet		inlet		outlet		inlet		outlet	
searches	10.15 am	am	Pressure		Pressure		Temp		Temp		Temp		Temp	
in En-	10.30		P1 bar	0.44	P3 bar	11	T1 ? C		T2 ? C		T3? C		T4? C	
gineer-			0.61	0.68	11.5	12.4	30 31 33		56 59 63		48 49 53		22 23 25	
ing														
of	10.45		0.78		12.8		34		65		58		27	
Global	am	11.00 am	0.98	1.02	14	15.5	35 35 35		68 70 73		59 57 58		30 32 33	
Jour-	11.15 am	11.30	1.02		15.5									
nal														
	am													
	11.45		1.02		15.5		35		75		62		33	
	am													
	12.00		1.09		16		34		77		62		32	
	Pm													
	12.15		1.09		16		34		77		61			
	pm													

Figure 10: Table 2 :

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Figure 11: Table 3 :

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