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1	The Characteristics of Torsionally Flexible Metal Coupling
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## 6 Abstract

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<sup>7</sup> The paper presents construction I.metal couplingIntroduction metal coupling of high torsional
<sup>8</sup> flexibility. In addition, the paper presents preliminary tests results which enable to determine

- <sup>9</sup> the above characteristics
- 11 *Index terms* machines, construction of machines, driving device, couplings II.

he tendency to increase the effectiveness of technological processes and at the same time the efficiency of machines through growth of power with simultaneous optimisation of their size and mass often leads to the increase of dynamic interactions of particular units of the machines. The dynamic interactions taking place in the power transmission system components are definitely disadvantageous. Broad research has shown that even relatively small dynamically affecting force can cause considerably bigger internal forces and displacements to appear than significantly bigger statically affecting force.

From the experiments regarding exploitation and the information of machine shops stems the fact that the 18 19 level of knowledge about constructing and the insufficiently examined problems with the process of utilization 20 need further research. The efficacy and reliability of machines are shaped during the processes of construction, 21 manufacturing and exploitation. Sufficient knowledge about working load affects the decisions made at all three of those phases and that has a close link to the durability of particular elements and units of the machines. In the 22 23 process of construction nominal external load is taken into account with a certain reserve, e.g. with the aid of the factor of implementation K A (formerly overload K p). However, assessing other factors that have influence on 24 the load of the elements of the machine, like e.g. those stemming from faulty performance and assembly as well 25 as the interaction between cooperating elements, is hampered because of their random character and unspecified 26 synergism in generating dynamic loads. 27

Author : Prof. nzw. Institute of Mining Mechanisation, Faculty of Mining and Geology at the Silesian University of Technology Akademicka 2, 44-100 Gliwice, Poland. E-mail : krzysztof.filipowicz@polsl.pl Decreasing the dynamic loads between particular elements of the power transmission system components of a mill, which usually include engine, mechanical gear and actuator can be accomplished through the accurate selection of couplings linking those respective elements. Thanks to couplings it is possible to considerably reduce the dynamic loads in the power transmission system components caused by both external and internal factors. This function is best fulfilled by torsionally flexible couplings characterised by adequate resilient and deadening features.

In the majority of the machines used in mining, building and road industry most of the power transmission 35 system components is liable to random loads both in the phase of start and steady flow as well as stop. Those 36 power systems are prone to be affected by overloads, shock loads and frequent starting under great load. An 37 excellent instance of the above is the power system of a drag chain conveyor. Difficult operating conditions 38 require the usage of couplings with great flexibility susceptibility. Currently the most loaded power transmission 39 40 systems of mining machines employ one or two liner flexible couplings, liquid couplings of various kinds (e.g. 41 SH, TV produced by Voith Turbo GmbH, etc.) or different types of integrated power transmission systems, like 42 WB/CST DBT, that comprise of epicyclic gearbox and multiple-plate coupling. The aforementioned examples of couplings are characterised by specific advantages and disadvantages described in technical literature. 43

The Institute of Mining Mechanisation developed a completely new construction, namely torsionally flexible metal coupling, which fulfils most of the requirements of application as regards the considerably loaded power transmission systems of mills ??1, ??, ??, ??].

Through the mitigation of torque alterations this coupling causes the absorption of energy, which affects the stabilisation of torsional vibration and the beneficial change of its flow. This contributes to a significant reduction 49 of dynamic loads and thereby ensures the accurate performance of the power transmission system of a mill. And 50 this means the increase of durability and reliability of its constituents.

## <sup>51</sup> 1 The Characteristics of Torsionally Flexible Metal Coupling

52 Dr. Hab. In?. Krzysztof Filipowicz

One of the constructional forms of bidirectional torsionally flexible metal coupling is shown in figure 1. The tenet of torsionally flexible metal coupling rests in the operating torque affecting the active part of the coupling directly through the shaft (1) and then being conveyed to the sliding sleeve (2) with the aid of a multi-start thread. The increasing value of the torque causes the rotation of the shaft (1) against the sliding sleeve (2) and at the same time the torque converter housing (3). The axial force that is thus produced in the multi-start thread initiates plane motion of the sliding sleeve alongside the crankshaft centreline (release lever axle).

The limitation of the motion of the sliding sleeve to plane motion is achieved through motor splined coupling positioned between the sleeve (2) and the torque converter housing (3). Plane motion of the sliding sleeve causes simultaneously the compression of a set of dished plate springs (4) that is especially selected to match the assumed qualities of the coupling. The compression of the springs causes the internal elastic strain force to affect the set. This force in every temporal, fixed location of the sliding sleeve counterbalances the axial force produced in the multistart thread, which is the result of the external insert momentum operating. Every temporal overload of the power system with the insert momentum triggers additional compression of the resilient elements of the coupling

66 and decreasing the load causes their annealing.

The static characteristic of a flexible coupling is a dependency of the torque M obr , which turns the coupling by the specific angle of rotation ? between the active and the passive element of the coupling. The changes of the torque in this case should be very slow.

## 70 2 M obr = f (?)

71 (1)

Where: M obr -The torque moment on the flexible coupling, Nm, ? -Relative angle of rotation of the coupling
radian or degrees.

The shape of the static characteristics depends on the construction of the coupling, the material used for the flexible connecting link i.e. the elastic-damping system and the very shape of the connecting link. In case of couplings with flexible rubber elements or elastomers which are most commonly used in power transmission systems of working machines, the characteristic is nonlinear with damping which makes it hard to choose the proper coupling for the power transmission system.

a) The testing stand and methodology of determining the static characteristics of a flexible coupling

The testing stand used to determine the static characteristics of a coupling is used to test the mechanical couplings built in the Institute of Mining Mechanisation at the Silesian University of Technology, as shown in Fig. 2.

The electric motor (1) powered by the inverter, which enables the smooth regulation of the rotational speed from 0 to 1700 min -1, is connected with the tensometric torque meter (5) used to test the torque moment M obr = M stat. The value of the measured torque is read and recorded by means of a computerised measuring apparatus (7) of type SCXI produced by National Instruments. One of the elements of the tested flexible metal coupling -the housing (??) is directly attached to the output shaft of the torque meter. The other element of the coupling -the output shaft -is attached to the hydraulic disc brake (3) which is controlled from the pressure supply system (4).

The output shaft of the coupling i.e. the passive element of the coupling is attached to the rotary disc with an angular scale  $0^{\circ}$ ÷360° (??). An indicating needle attached to the coupling housing was used to read the relative angular displacement of the elements ? = ? stat .

When the coupling is secured at the testing stand by means of a fully blocked hydraulic brake, one of the 93 elements of the flexible clutch may no longer turn. The electric motor is activated by means of an inverter and 94 the static torque M stat is gradually increased. The torque moment changes every 10 Nm in order to reach the 95 maximum M max = 100Nm. At the same time the relative angular displacement of the coupling elements? stat 96 on the angular scale of the disc is read by means of an indicating needle attached to the movable housing. The 97 precision of the reading is  $\pm 1^{\circ}$ . The readings of the dependency of the torque M stat on the relative angular 98 displacement ? stat is carried out with and without the load. The aim of this procedure is to determine the 99 histeresis loop which presents the value of the mechanical energy damping in the coupling. The tests were carried 100 out with the use of couplings with four different sets of disc springs. The choice of these sets was based on the 101 102 analysis of disc springs characteristics arranged in packs and forming sets. The sets were chosen so that with the 103 maximum torque moment M max = 100 Nm, the springs would work below the accepted working range, that is 75% of the maximum deflection. 104

105 Table 1 contains the data regarding the disc springs used in the coupling and the sets that were placed in it.

() When the coupling characteristics are linear as those obtained in the tests, we can determine the coefficient
called the torsional rigidity of the coupling k for each of them. For the determined linear static characteristics,
the dependency is expressed as follows:

109 110 (2)

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displacement of the coupling elements, radian or degrees.

Figure 1: Figure 1 :

Where : M stat -the static torque moment on the coupling, Nm ? stat -the corresponding relative angular splacement of the coupling elements radian or degrees  $^{-1}$ 

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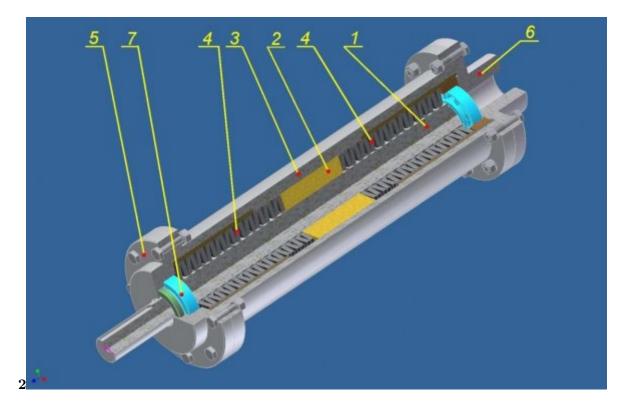


Figure 2: Figure 2 :

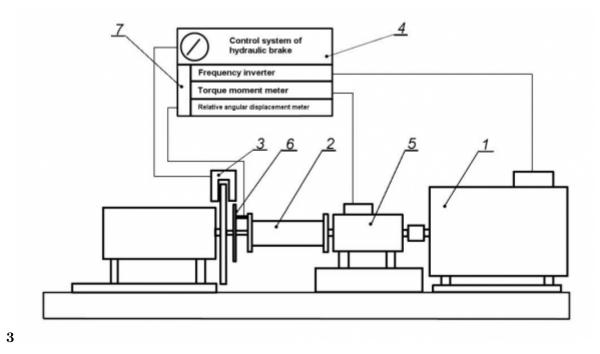


Figure 3: Figure 3 :

Coupling Type of disc springs DIN 2093		The number	The arrangement of the
Num-		of springs in	pack
ber		the pack	
1	40 x 20,4 x 2,5	5	$\ll \ll \gg \gg >$
2	$40 \ge 20,4 \ge 2,5$	4	«« »»
3	$40 \ge 20,4 \ge 2,25$	4	«« »»
4	$40 \ge 20,4 \ge 2$	4	«« »»

[Note: A Year]

Figure 4: Table 1 :

- 112 .1 ()
- 113 A Year ()

## 114 .2 A Year

The flexible metal coupling is not only rigid but it also absorbs energy. The histeresis loop was obtained during loading and unloading and it presented the histeresis losses of the mechanical energy in the coupling called damping. In the flexible metal coupling damping is achieved by means of constructive friction in screw joints mainly.

The value of the mechanical energy damping in the coupling is specified by the damping coefficient that is determined as follows:

121 (3) Where:

A r -damping during one working cycle, Nm ? degrees. A s -elastic strain during one work cycle, Nm ? degrees. The obtained torsional rigidity coefficients as well as the damping coefficient for the tested options of the torsionally flexible metal coupling are presented in Table **??**. The preliminary tests carried out to determine the static characteristics of a torsionally flexible metal coupling lead to the following general conclusions:

? The proposed testing stands as well as the methodology of the static characteristics test enable to quickly 126 and correctly determine the static characteristics with acceptable precision and repeatability of the achieved 127 measurements results. ? The determined static characteristics of the torsionally flexible coupling are linear with 128 damping of linear correlation coefficient r = 0.98. ? The position and slope of the static characteristics and 129 values of the torsional rigidity and damping coefficients are strictly linked with the application of the disc springs 130 sets in the coupling, the more rigid the springs the greater torsional rigidity of the coupling and the lower the 131 damping coefficient. ? The coupling characteristics can be shaped by the proper choice of disc springs sets, 132 though one has to take into consideration the condition of the proper performance of the spring i.e. up to 75% of 133 its maximum strain. ? The torsionally flexible metal coupling enables to obtain significant angles of the relative 134 angular displacement of the elements even up to a few hundred degrees; the obtained angles are determined 135 by the applied sets of disc springs. The presented conclusions clearly indicate the advantageous features of the 136 presented constructive solution of the flexible coupling. None of the mechanical couplings which have been used 137 so far in road, building and mining machinery has the above characteristics and the static characteristics as those 138 mentioned above. Classic flexible couplings enable to obtain angles of mutual torsion only up to a few degrees, 139 in most cases with difficult to describe nonlinear characteristics. 140