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Experimental Investigations on a Four Stroke Compression Ignition Engine using Neat Tobacco Seed Oil as an Alternate Fuel

M Sadanandam¹
 ¹ Kakatiya University
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7 Abstract

8 Performance, emission and combustion tests were carried out on a four stroke compression

⁹ ignition engine using tobacco seed oil by varying fuel injection pressure and fuel injection

¹⁰ timing and compared with base line diesel. The main objective of conducting the performance

¹¹ test on a most widely used agricultural segment engine with tobacco seed oil to help the

¹² farming community to use tobacco seed oil in case of emergency and short term applications.

13 At 260 Bar and 26° BTDC the performance of the engine is quite encouraging compared to

the operation of the engine at standard injection pressure of 205 bar and 23° BTDC

15

Index terms— compressed air motor (cam)/ pneumatic wrench, compressed air technology, ecofriendly,
 global conditions, renewable energy handling

18 1 Introduction

he depletion of fossil fuels all over the world is pressing the people to search for alternative fuels. Mostly the 19 20 research is going on tree based oil seeds for the extraction of oil and conversion of these oils as bio diesel for the purpose of IC engine applications. The tree based oil seed production takes longer time compared to plant 21 based oil seed production. Countries like Argentina, China, Brazil, Bulgaria, Greece, India, Indonesia, Turkey, 22 Tunisia, etc are cultivating tobacco for commercial purpose. In majority of the tobacco cultivation process only 23 the tobacco leaves are used and the tobacco seeds are unused (1,2,3). Due to the worldwide tobacco abuse the 24 crop cultivation is coming down. If the tobacco seeds which are left out in the tobacco fields are used for tobacco 25 seed oil extraction then it can be used for many useful applications. emergency and short term application for 26 the agricultural segment engines like water pumps, diesel generators, power tillers etc. In some of the tobacco 27 growing countries due to the shortage of cooking oil the tobacco seed oil is being used as edible oil after due 28 processing of this oil after removal of toxic contents as it contains omega-3 fatty acid which is essential for human 29 body (1). 30

The oil cake emerging from the tobacco seed oil extraction can be used as a good manure for the agricultural fields moreover the fuel import bills will come down and this will strengthen any nation for its energy self sufficiency. For this purpose if the engine is tuned by varying the fuel injection pressure and timing to yield best performance such that the unused tobacco seeds can be converted to most useful fuel.

In the present experimental investigation the engine performance, emission and combustion characteristics using tobacco seed oil as fuel at fuel injection pressures of 205, 220, 240 and 260 bar and fuel injection timing of 23°, 26°, 28° BTDC were evaluated and compared with base line diesel operation.

³⁸ 2 II. Characterisation of Tobacco Seed Oil

³⁹ The details of tobacco plant and properties of tobacco seed oil are given below.

$_{40}$ 3 a) Tobacco Plant

Initially the tobacco plants are raised from tobacco beds as is done in the case of paddy plantation. The small tobacco plants are then planted in an array and frequently irrigated. The tobacco plant grows from 1 meter to 43 2 meters in height and after attaining maximum height beautiful pink flowers will emerge. These flowers will 44 turn to green capsules containing numerous tobacco seeds. When these green capsules turn to brown colour after

considerable drying the tobacco seeds will be ready for oil extraction and the oil yield will be around 25 to 30%.

⁴⁶ 4 b) Tobacco seed oil

The tobacco seed oil in sufficient quantities were procured from Sri Laxmi venkatesh wara oil mill, Santhanuthalapadu, Prakasham district, Andhra Pradesh. The fatty acid composition of tobacco seed oil is shown in

49 5 Experimental Setup

50 6 Experimental Procedure

A four stroke single cylinder direct injection water cooled computerized Kirloskar TV1 diesel engine shown in fig. 1 is used for testing the performance by enhancing fuel injection pressure and fuel injection advance. After ensuring no load on the eddy current dynamometer the engine was started on base line diesel and performance, emission and combustion tests were conducted using base line diesel as fuel at manufacturers recommended fuel injection pressure of 205 Bar and fuel injection timing of 23° BTDC. The performance, emission and combustion characteristics are plotted by varying 20, 40, 60, 80 and 100% load with base line diesel operation.

The performance emission and combustion tests were conducted using tobacco seed oil following the above 57 procedure to evaluate the performance and emission analysis at different injection pressures (205, 220, 240 and 58 59 260 bar) and keeping the fuel injection timing at 23° BTDC. Before introducing tobacco seed oil as test fuel, the engine was run on base line diesel for 10 to 15 minutes and after stabilizing the engine parameters, the tobacco 60 seed oil was introduced. After running the engine for 10-15 minutes on the tobacco seed oil and stabilizing the 61 parameters with the tobacco seed oil the readings were recorded and after completion of the tests with tobacco 62 seed oil the engine was switched over to base line diesel and run for 10 to 15 minutes on base line diesel before 63 stopping the ()A Year 013 2 64

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66 Oil as an Alternate Fuel engine to avoid cold starting and fuel injector and fuel pump plunger sticking problems.

The above test procedure is repeated by keeping the fuel injection timing at 26° and 28° BTDC and the fuel

⁶⁸ injection pressure at 205, 220, 240 and 260 Bar.

69 Uncertainity Analysis:

The experimental investigations involve the use of different instruments for measurement of different 70 parameters. These instruments or equipment are made by different manufacturers using different technologies. 71 The accuracy of measurement and their performance may vary depending on the operating and experimental 72 environment. Hence the uncertainty occurs due to fixed or random errors. The uncertainty in the measured 73 parameter can be estimated based on analytical methods. Thus the uncertainty can be estimated using Gaussian 74 distribution method with confidence limits of 2? Using the above equation for a given operating condition the 75 uncertainties are computed for the measured quantities are listed below N= Speed \pm 1.0 % W= Load \pm 0.5 % 76 BP= Brake Power \pm 0.5 % BSFC= Brake specific fuel consumption \pm 1.4 % EGT= Exhaust gas temperature 77 \pm 0.2 % BTE = Brake thermal efficiency \pm 1. 78

79 7 Results & Discussions

80 The variation of brake specific energy consumption of tobacco seed oil with respect to brake power is presented 81 in Fig. 2. For both diesel and tobacco seed oil operation the brake specific energy consumption was decreased with increase of brake power up to 80% load and at this point the lowest brake specific energy consumption for 82 diesel, tobacco seed oil at best injection pressure and timing and tobacco seed oil at standard setting of injection 83 pressure and timing are 12837.82, 14245.58, 16479.86 kJ/kWh respectively. Hence 19% saving in brake specific 84 energy consumption with tobacco seed oil can be achieved by best injection pressure and timing over standard 85 setting of injection pressure and timing. Fig. ??. shows the variation of exhaust gas temperature with brake 86 power. The exhaust gas temperature with tobacco seed oil is lower than diesel operation. This is due to higher 87 heating value of diesel fuel. However with tobacco seed oil at standard setting of injection pressure and timing 88 the exhaust gas temperatures are very close to diesel operation indicates the ineffective combustion due to lower 89 injection pressure which leads to after burning. 90

91 The variation of brake thermal efficiency with brake power is shown in Fig. ??. The highest brake thermal 92 efficiency is obtained at around 4 kW with both diesel and tobacco seed oil. Throughout the operating range the 93 brake thermal efficiency is higher with diesel compared to tobacco seed oil. The peak brake thermal efficiency 94 with diesel, tobacco seed oil at best setting and tobacco seed oil at standard setting of injection pressure and timing are 28.05%, 24.92%, 21.54% respectively. The decreased values of brake thermal efficiency with tobacco 95 seed oil over diesel indicates lower heating value and poor combustion due to high viscosity of tobacco seed oil. 96 However with best setting of fuel injection pressure and timing there is 15.7% increase in brake thermal efficiency 97 over standard setting with tobacco seed oil. This improvement in brake thermal efficiency can be attributed to 98

⁹⁹ improved combustion due to higher injection pressure and advanced injection timing.

Concentration of unburnt hydrocarbon emission variation with brake power is represented in Fig. ??. Higher values of unburnt hydrocarbon with tobacco seed oil over diesel indicates improper combustion of tobacco seed oil due to more heterogeneous mixture formation resulting from higher viscosity and low volatility.

Emission of carbon monoxide variation with brake power is indicated in Fig. ??. The carbon monoxide emissions are very low in all the cases as expected in any of the compression ignition engines due to the presence of excess air. However there is an indication of slightly higher values of carbon monoxide with tobacco seed oil over diesel operation.

The variation of Oxides of Nitrogen emissions with brake power is shown in Fig. ??. There is higher Oxides of Nitrogen concentration in the exhaust of tobacco seed oil operation when compared to diesel. This is obvious due to the more availability of oxygen with tobacco seed oil as the tobacco seed oil itself contains oxygen in its molecular structure. However with tobacco seed oil at best injection pressure and timing the Oxides of Nitrogen emissions are higher over that of at standard setting due to prevailing of higher combustion temperatures. When the engine was operated on tobacco seed oil at best setting of injection pressure and timing at 80% load Oxides of Nitrogen emissions were 425 ppm and that of base line diesel operation were 126 ppm.

At standard setting of injection pressure and timing Oxides of Nitrogen were 299 ppm. Fig. ??. shows the variation of smoke opacity with brake power. The smoke intensity is higher with tobacco seed oil as compared to diesel due to higher viscosity of tobacco seed oil leading to thermal cracking. At 80% load when the engine operated on tobacco seed oil at best setting of injection pressure and timing smoke level is 46% lower compared to the tobacco seed oil operation at standard setting of injection pressure and timing.

Fig. ?? explains pressure versus Crank Angle data pertaining to tobacco seed oil at standard and best injection pressure at 80% load. The effect of increase in fuel injection pressure from 205 bar to 260 bar and advancing the fuel injection timing from 23° BTDC to 26° BTDC improves the peak pressure from 65.38 bar to 74.9 bar. This may be due to the decrease in ignition delay as a result of enhanced injection pressure and advanced fuel injection timing.

Fig. 10 indicates the rate of pressure rise versus Crank Angle and when the injection pressure is () A Year increased the maximum rate of pressure rise increased from 3.77 bar /°Crank Angle to 4.85 bar/°Crank Angle. Fig. 11 shows the highest net heat release rate which increases from 35.9 Joules /°Crank Angle to

44.11 Joules/°Crank Angle indicating the improvement of net heat release rate with the increase in fuel injection
pressure and advanced fuel injection timing.

The cumulative heat release is shown in Fig. 12 There is an improvement of highest cumulative heat release from 1.14 kJ at 509 °Crank Angle to 1.17 kJ at 423 ° Crank Angle. For diesel the highest cumulative heat release value of 0.91 kJ at 391°Crank Angle was observed.

The mass fraction burnt in % is shown in Fig. 13 The 5% mass fraction burnt for diesel, tobacco seed oil at standard and best settings are 354, 358 and 354°Crank Angle respectively. The 90% mass fraction burnt for diesel, tobacco seed oil at standard and best settings are 375, 381 and 379°Crank Angle respectively. The occurrence of 5% and 90% mass fraction burnt with best setting of injection pressure and timing has been advanced by 4° and 2° crank Angle compared to standard setting of injection pressure and timing. V.

137 8 Conclusions

The tobacco seed oil can be used as an alternate fuel in cases of emergency and short term use with proper 138 precautions and tuning of the engine for best performance level. When the engine is operated with tobacco seed 139 oil at best injection pressure and timing of 260 bar and 26° BTDC over standard setting of injection pressure 140 and timing of 205 bar and 23° BTDC, 19% decrease in brake specific fuel consumption, 15.7% increase in brake 141 thermal efficiency at 80% load and lower emissions were observed. The combustion data reveals improvement in 142 peak pressure from 65.38 bar to 74.9 bar, maximum rate of pressure rise from 3.77 bar /°Crank Angle to 4.85 143 bar/°Crank Angle, highest net heat release rate from 35.9 Joules /°Crank Angle to 44.11 Joules/°Crank Angle, 144 highest cumulative heat release from 1.14 kJ to 1.17 kJ and mass fraction burnt parameters. Hence for this 145 particular engine 260 bar fuel injection pressure and 26° BTDC fuel injection timing is a best option to run with 146 neat tobacco seed oil for emergency and short term applications. 147

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Figure 1: Figure 1 :

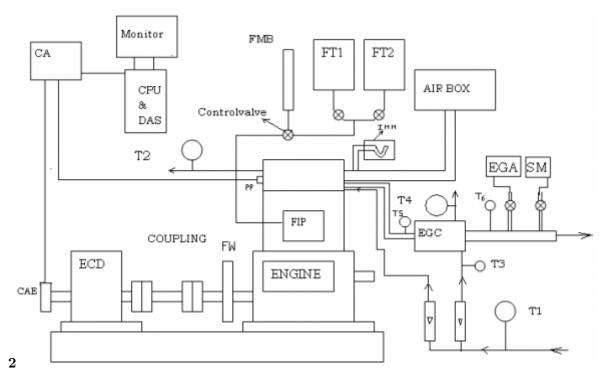


Figure 2: Figure 2 :

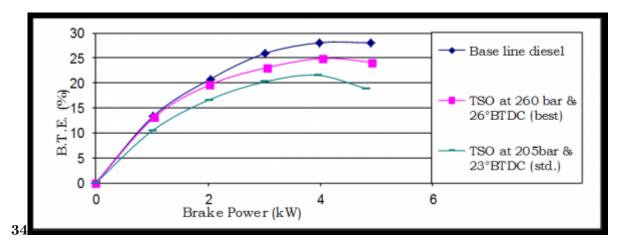


Figure 3: Figure 3 :Figure 4 :Experimental

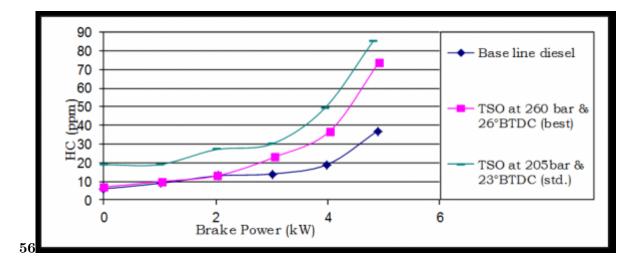


Figure 4: Figure 5 : Figure 6 :

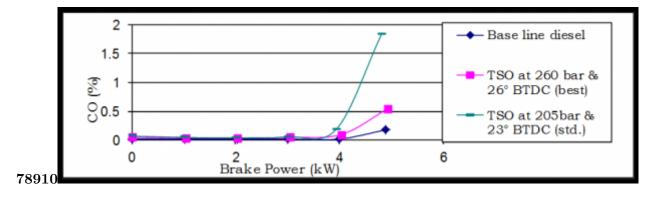


Figure 5: Figure 7 : Figure 8 : Figure 9 : Figure 10 :

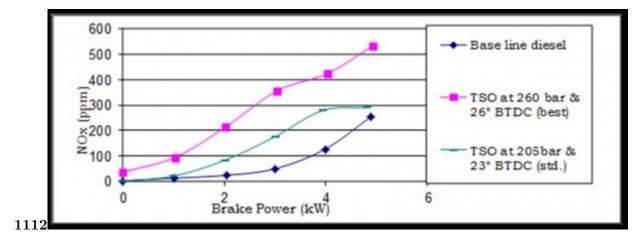


Figure 6: Figure 11 : Figure 12 :

 $\mathbf{2}$

Figure 7: Table 2 .

	Air measurement manometer							
13	Made				MX 201			
14	Type				U-type			
15	Range				100-0-100mm			
				Eddy curren	t dynamometer			
16	Model				AG-10			
17	Type				Eddy current			
18	Maximum				7.5 kW at $1500-3000 rpm$			
IV.					CO= carbon Monoxide \pm 0.2 %			
					HC= Hydro carbon \pm 0.2 % NOx= Oxides of Nitrogen \pm 0.3 % SO= Smoke opac			
S.No) Instruments	5GAS	ANALISER	CO (0-15%)	$PP = Pressure pickup \pm 1.0 \%$ HC (0-30000PPM) NOx (0-5000PPM) O2 (0-25%) CO			

- S.No Instruments 5GAS ANALISER
- 1 2 SMOKE METER LOAD INDI-
- 3 CATOR

4 PRESSURE PICKUP PRESSURE (0-110Bar)

5 6 CRANK ANGLE ENCODER EXHAUST GAS TEMPERATU RE INDICATOR SPEED FUEL MEA S.NO

- 78
- $1 \ 2$
- 3
- 4 Nozzle opening pressure Governor type 5No of cylinders 6
- No of strokes 7
- 8 Fuel
- 9 Rated power
- Cylinder diameter 10
- 11 Stroke length
- 12Compression ratio

200-225 bar Mechanical centrifugal type Single cylinder Four stroke H.S.diesel $5.2 \mathrm{kW}(7 \mathrm{HP}) @1500 \mathrm{rpm}$ $87.5\mathrm{mm}$ $110 \mathrm{mm}$ 17.5P:1

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Stearic (18:0) Oleic (18:1) Linoleic (18:2) Linolenic (18:3) 10.96-8.87 3.34-3.49 14.54-12.4 69.49-67.75 0.69-4

Properties Density	Tobacco seed oil 0.915 24 230 248 38438 0.45	Base
at 32° C in gm/cc		line
Kinematic Viscosity		diesel
at 32 °C in C s		0.82
T Flash point ° C		4
Fire point Carbon		52
residue Calorific		57
value k (%)		42000
		0.1

J/kg

[Note: © 2013 Global Journals Inc. (US) Global Journal of Researches in Engineering Volume XIII Issue IV Version I 8 () A Year 013 2]

Figure 9: Table 3 :

1

Figure 10: Table 1

 $\mathbf{2}$

Figure 11: Table 2

148 .1 Acknowledgements

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