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Pole Placement Approach for Controlling Double Inverted Pendulum

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Abstract - In this paper, we present in-depth analysis of the classical double inverted pendulum (DIP) system using the DIP modeling and the pole placement approach to control it. The double inverted pendulum system has the characteristics of multiple variables, non-linear, absolute instability; it can reflect many key issues in the progress of control, such as stabilization, non-linear and robust problems etc. DIP model is a simplified model of the anterior-posterior motion of a standing human. DIP has four equilibrium points (Down-Down, Down-Up, Up-Down, Up-Up). The objective of this paper is to keep the double pendulum in an Up-Up unstable equilibrium point. Modeling is based on the Euler-Lagrange equations, and the resulted non-linear model is linearized around Up-Up position. The built of mathematical model of double inverted pendulum plays a guiding role on the stability of the system. The eigen-values of the system which are the poles of the system have enormous influenced on stability and system response. Pole placement is the control method which places the poles at the desired position to control the system by calculating gain matrix of the system.

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Pole Placement Approach for Controlling Double Inverted Pendulum

Nita H. Shah ^α & Mahesh Yeolekar ^σ

Abstract - In this paper, we present in-depth analysis of the classical double inverted pendulum (DIP) system using the DIP modeling and the pole placement approach to control it. The double inverted pendulum system has the characteristics of multiple variables, non-linear, absolute instability; it can reflect many key issues in the progress of control, such as stabilization, non-linear and robust problems etc. DIP model is a simplified model of the anterior-posterior motion of a standing human. DIP has four equilibrium points (Down-Down, Down-Up, Up-Down, Up-Up). The objective of this paper is to keep the double pendulum in an Up-Up unstable equilibrium point. Modeling is based on the Euler-Lagrange equations, and the resulted non-linear model is linearized around Up-Up position. The built of mathematical model of double inverted pendulum plays a guiding role on the stability of the system. The eigen-values of the system which are the poles of the system have enormous influenced on stability and system response. Pole placement is the control method which places the poles at the desired position to control the system by calculating gain matrix of the system. In this paper, the performance of the pole placement method is analyzed by MATLAB to control the double inverted pendulum.

Keywords : double inverted pendulum; linear time-invariant system; pole placement method.

I. INTRODUCTION

The study of humanoid robot is currently one of the most exciting research projects. Even if some of those works have already demonstrated very reliable dynamic biped walking (Yamaguchi, Soga, Inoue & Takanishi, 1999; Hirai, Hirose, Haikawa & Takenaka, 1998; Nishiwaki, Sugihara, Kagami, Kanehiro, Inaba & Inoue, 2000), we believe it is still important to understand the mathematical theoretical background of biped locomotion. In standing, it has become common to consider the body as an (single\double\triple) inverted pendulum pivoted at the ankles. Moreover, up ride of a human shoulder is also considered as a motion of an (single\double\ triple) inverted pendulum (Jadlovská, 2011; Jadlovská & Jadlovská, 2010).

An inverted pendulum system is a typically non-linear, redundancy, uncertainty, strong coupling and natural characteristics of instabilities. All these features make it the ideal model of advanced control theory and typical experiment platform of test control results. There

are a number of different kinds of the inverted pendulum systems presenting a variety of control challenges. The most common types are the single inverted pendulum on a cart (Ohsumi & Izumikawa, 1995; Åström & Furuta, 2000), the double inverted pendulum on a cart (Zhong & Rock, 2001), the double inverted pendulum with an actuator at the first joint only (Pendubot) (Graichen & Zeitz, 2005), the double inverted pendulum with an actuator at the second joint only (Acrobot) (Hauser & Murray, 1990), the light weight rotary pendulum (Brockett & Hongyi, 2003).

In this paper, we have addressed the problem of stability of double inverted pendulum in the upright position using the pole placement method. For this, we have assumed that the double inverted pendulum is pivoted at the lower end of inner arm (see figure 1). The first step to achieve the objective is to understand the dynamics of the system of double inverted pendulum by developing the mathematical modelling of the system. In modelling, we have used Euler-Lagrange formulation to find equation of motion. In the second step, we linearized this non-linear system of double inverted pendulum in the up-up position and build up its linear state space model. The linearization is one of the most important issues for control of non-linear systems. There are lots of studies in the literature regarding linearization (Jordan, 2006; Wang, Chen & Zhou, 2000; Conga, Wanga & Hill, 2005). In the next step, the stability and controllability criteria showed that the system is unstable but it is controllable.

To control this unstable system, we have employed the pole placement method. In this method, the poles are the eigen-values of linear state space model and the calculated gain matrix places the poles at desired position to stabilize a system. In the simulation part of this paper, numerical and graphical simulations for control task are given to show the effectiveness of the proposed pole placement scheme.

II. MATHEMATICAL MODELING OF DIP

In this section, we will describe mathematical model necessary for stability and controllability analysis. The mechanism of the double inverted pendulum is shown in Figure 1 schematically. The mathematical model of DIP can be derived using the Euler-Lagrange equation. The form of the Euler-Lagrangian equation used here is:

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$$\frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}} - \frac{\partial L}{\partial \theta} = \tau \quad (1)$$

Where $L = T - V$ is a Lagrangian, T is kinetic energy, V is potential energy, $\tau = [\tau_1 \ \tau_2]^T$ is

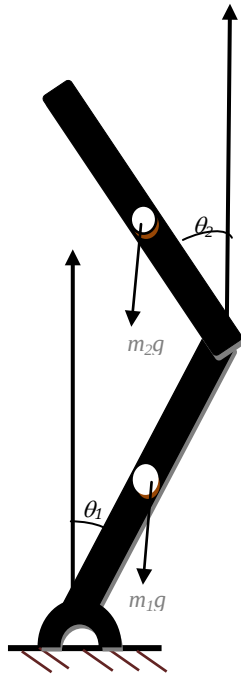


Figure 1 : Double Inverted Pendulum

the input generalized force vector produced by two actuators at the lower joint (ankle) and second at joint between to arm (knee), $\theta = [\theta_1 \ \theta_2]^T$ is generalized coordinate vector where θ_1 and θ_2 are angular positions of first arm, and second arm of the double pendulum. The kinetic and potential energies in terms of generalized coordinates can be determined as:

$$T = \begin{bmatrix} \frac{1}{2} (m_1 l_1^2 + I_1) \dot{\theta}_1^2 \\ \frac{1}{2} [m_2 (4l_1^2 \dot{\theta}_1^2 + 4l_1 l_2 \cos \theta_2 \dot{\theta}_1 \dot{\theta}_2 + l_2^2 \dot{\theta}_2^2) + I_2 \dot{\theta}_2^2] \end{bmatrix} \quad (2)$$

$$V = \begin{bmatrix} m_1 g l_1 \cos \theta_1 \\ m_2 g (2l_1 \cos \theta_1 + l_2 \cos \theta_2) \end{bmatrix} \quad (3)$$

Differentiating the Lagrangian $L = T - V$ by generalized coordinate's vector yields Euler-Lagrange equation (1) as:

$$\begin{aligned} & [m_1 l_1^2 + I_1 + 4m_2 l_1^2 + 4m_2 l_1 l_2 \cos \theta_2 + m_2 l_2^2] \ddot{\theta}_1 - [2m_2 l_1 l_2 \cos \theta_2 + m_2 l_2^2] \ddot{\theta}_2 + 2m_2 l_1 l_2 \sin \theta_2 \dot{\theta}_2^2 - 4m_2 l_1 l_2 \sin \theta_2 \dot{\theta}_1 \dot{\theta}_2 \\ & - (m_1 + 2m_2) g l_1 \sin \theta_1 - m_2 g l_2 \sin(\theta_1 - \theta_2) = \tau_1 \end{aligned}$$

$$[-2m_2 l_1 l_2 \cos \theta_2 - m_2 l_2^2] \ddot{\theta}_1 + [m_2 l_2^2 + I_2] \ddot{\theta}_2 + 2m_2 l_1 l_2 \sin \theta_2 \dot{\theta}_1^2 + m_2 g l_2 \sin(\theta_1 - \theta_2) = \tau_2$$

The matrix form of the system is

$$\begin{bmatrix} m_1 l_1^2 + I_1 + 4m_2 l_1^2 + 4m_2 l_1 l_2 \cos \theta_2 + m_2 l_2^2 & -2m_2 l_1 l_2 \cos \theta_2 - m_2 l_2^2 \\ -2m_2 l_1 l_2 \cos \theta_2 - m_2 l_2^2 & m_2 l_2^2 + I_2 \end{bmatrix} \begin{bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{bmatrix} + \begin{bmatrix} 0 & 2m_2 l_1 l_2 \sin \theta_2 \\ 2m_2 l_1 l_2 \sin \theta_2 & 0 \end{bmatrix} \begin{bmatrix} \dot{\theta}_1^2 \\ \dot{\theta}_2^2 \end{bmatrix} + \begin{bmatrix} -4m_2 l_1 l_2 \sin \theta_2 \dot{\theta}_1 \dot{\theta}_2 - (m_1 + 2m_2) g l_1 \sin \theta_1 - m_2 g l_2 \sin(\theta_1 - \theta_2) \\ m_2 g l_2 \sin(\theta_1 - \theta_2) \end{bmatrix} = \begin{bmatrix} \tau_1 \\ \tau_2 \end{bmatrix}$$

a) Linearisation of the System

The tracking controller of DIP is designed using the Gain Scheduling method based on the linearisation of the system equations around certain equilibrium points. In this paper, we linearize the system at the vertical unstable equilibrium by taking.

$$\theta_1 \approx \theta_2 \approx 0,$$

$$\cos \theta_1 \approx \cos \theta_2 \approx 1$$

$$\sin \theta_1 \approx \theta_1; \sin \theta_2 \approx \theta_2;$$

$$\theta_1 - \theta_2 \approx 0; \cos(\theta_1 - \theta_2) \approx 1; \sin(\theta_1 - \theta_2) \approx \theta_1 - \theta_2$$

$$\dot{\theta}_1^2 \approx \dot{\theta}_2^2 \approx 0$$

So,

$$\begin{aligned} & \left[m_1 l_1^2 + I_1 + 4m_2 l_1^2 + 4m_2 l_1 l_2 + m_2 l_2^2 \right] \ddot{\theta}_1 + \left[-2m_2 l_1 l_2 - m_2 l_2^2 \right] \ddot{\theta}_2 \\ & - (m_1 + 2m_2) g l_1 \theta_1 - m_2 g l_2 (\theta_1 - \theta_2) = \tau_1 \\ & \left[-2m_2 l_1 l_2 - m_2 l_2^2 \right] \ddot{\theta}_1 + \left[m_2 l_2^2 + I_2 \right] \ddot{\theta}_2 + m_2 g l_2 (\theta_1 - \theta_2) = \tau_2 \end{aligned}$$

The matrix of the linear system

$$\begin{bmatrix} m_1 l_1^2 + I_1 + 4m_2 l_1^2 + 4m_2 l_1 l_2 + m_2 l_2^2 & -2m_2 l_1 l_2 - m_2 l_2^2 \\ -2m_2 l_1 l_2 - m_2 l_2^2 & m_2 l_2^2 + I_2 \end{bmatrix} \begin{bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{bmatrix} + \begin{bmatrix} -(m_1 + 2m_2) g l_1 - m_2 g l_2 & m_2 g l_2 \\ m_2 g l_2 & -m_2 g l_2 \end{bmatrix} \begin{bmatrix} \theta_1 \\ \theta_2 \end{bmatrix} = \begin{bmatrix} \tau_1 \\ \tau_2 \end{bmatrix}$$

b) State space model for the above linear system

The state space model equation for the system is

$$\begin{aligned} \dot{x} &= Ax + Bu \\ y &= Cx + Du \end{aligned} \quad (4)$$

where

$$A = M^{-1}N \quad B = M^{-1}T \quad C = I_{4 \times 4} \quad D = 4 \times 4 \quad u = \begin{bmatrix} \tau_1 \\ \tau_2 \end{bmatrix}$$

$$M = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & m_1 l_1^2 + I_1 + 4m_2 l_1^2 + 4m_2 l_1 l_2 + m_2 l_2^2 & -2m_2 l_1 l_2 - m_2 l_2^2 \\ 0 & 0 & -2m_2 l_1 l_2 - m_2 l_2^2 & m_2 l_2^2 + I_2 \end{bmatrix}$$

$$N = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ (m_1 + 2m_2) g l_1 + m_2 g l_2 & -m_2 g l_2 & 0 & 0 \\ -m_2 g l_2 & m_2 g l_2 & 0 & 0 \end{bmatrix}$$

$$T = \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ 1 & 0 \\ 0 & 1 \end{bmatrix}; \quad x = \begin{bmatrix} \theta_1 \\ \theta_2 \\ \dot{\theta}_1 \\ \dot{\theta}_2 \end{bmatrix}$$

c) Values of Parameters

The values of parameters for the given double inverted pendulum are assumed as follows:

m_1 = mass of inner arm = 0.4 kg

m_2 = mass of outer arm = 0.5 kg

l_1 = length of inner arm = 5 m

l_2 = length of outer arm = 5 m

g = gravitational acceleration = 9.8 m/s²

So the corresponding values of state space matrices are as follows:

$$A = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ -0.8276 & -1.4206 & 0 & 0 \\ -4.1012 & -2.1247 & 0 & 0 \end{bmatrix}$$

$$B = \begin{bmatrix} 0 & 0 \\ 0 & 0 \\ -0.0328 & -0.0908 \\ -0.0908 & -0.1775 \end{bmatrix}$$

III. STABILITY AND CONTROLLABILITY OF SYSTEM

a) Stability Criterion

A system (state space representation) is stable iff all the eigenvalues of the matrix A are inside the unit circle.

The eigen value of A of our system are: 0.0000 + 1.9939i, 0.0000 - 1.9939i, -1.0115, 1.0115 which are outside the unit circle because the modules of eigen values are greater than 1. So the system is unstable in absence of input force ($\tau_1 = 0$, $\tau_2 = 0$).

b) Controllability criterion

A system (state space representation) is controllable iff the controllable matrix $C = [B \ AB \ A^2B \ \dots \ A^{n-1}B]$ has rank n where n is the number of degrees of freedom of the system.

In our system, the controllable matrix $C = [B \ AB \ A^2B \ A^3B]$ has rank 4 which the degree of freedom of the system. So, the system is controllable.

If the system is controllable, then all set of distinct closed loop poles are assigned arbitrarily by output feedback gain (Kimura, 1975; Kimura, 1977; Wonham, 1967).

IV. POLE PLACEMENT

Block diagram of pole placement is given in Fig. 2. If the linearized system considered is completely state controllable, then poles of the closed-loop system may be placed at any desired locations by means of state feedback through an appropriate state feedback gain matrix K .

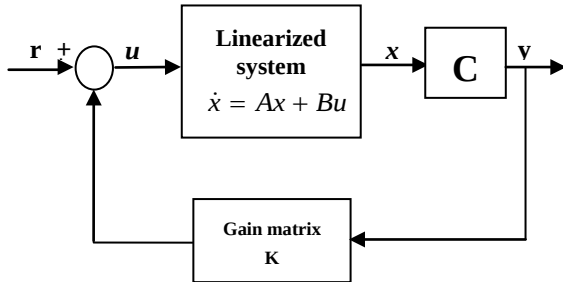


Figure 2: Block Diagram of Pole Placement

In this paper, we have used the following method to calculate state feedback gain matrix:

Pole placement with output feedback is displayed in Figure 2. In this paper, the reference signal, r , is taken zero. If an output feedback control

$$u = -Kx$$

is applied to (4), the closed-loop system becomes

$$\dot{x} = (A - KB)x$$

The poles assigned with output feedback are $\lambda = \{\lambda_1, \lambda_2, \lambda_3, \dots, \lambda_n\}$. The problem considered in this paper is finding gain matrix K for transferring the poles.

a) Gain Matrix Scheduling

Consider the controllability matrix $C = [B \ AB \ A^2B \ \dots \ A^{n-1}B]$ which is an $n \times pn$ order matrix. If the system is controllable, then the $\text{rank}(C) = n$. That means, it has only n linearly independent columns among the pn columns. Therefore, there will be many ways to construct an $n \times n$ similarity matrix which will give a multi-input controllable canonical form. In this paper, we use the following way:

Consider controllable matrix in n block as follows:

$$C = [b_1 \dots b_p : Ab_1 \dots Ab_p : \dots : A^{n-1}b_1 \dots A^{n-1}b_p]$$

Block 0 Block 1

Block $n-1$

Starting from the left in, this matrix, check each column, keeping count of the number of linearly independent columns we encounter. We may stop counting when n linearly independent columns are obtained.

Let the block in which we find the *last* (i.e., the n th) linearly independent column be denoted by the $(\mu-1)$ th block. Then the first block in which there are *no more* independent columns will be the μ th block. This μ is controllability index.

Rearranging these selected n linearly independent columns $b_1, b_2, \dots, b_p, Ab_1, Ab_2, \dots, A^{\mu-1}b_1, \dots$ we will get the invertible matrix M as:

$$M = [b_1 \dots A^{\mu_1-1}b_1 : b_2 \dots A^{\mu_2-1}b_2 : \dots : b_p \dots A^{\mu_p-1}b_p]$$

Where μ_i ($1 \leq i \leq p$) are the controllability indices of (A, B) .

The inverse of M is

$$M^{-1} = \begin{bmatrix} m_{11} \\ \vdots \\ m_{1\mu_1} \\ m_{21} \\ \vdots \\ m_{2\mu_2} \\ \vdots \\ m_{p1} \\ \vdots \\ m_{p\mu_p} \end{bmatrix}$$

Using this inverse matrix of M , calculate transformation matrix T as follows:

$$T = \begin{bmatrix} m_{1\mu_1} \\ \vdots \\ m_{1\mu_1} A^{\mu_1-1} \\ m_{2\mu_2} \\ \vdots \\ m_{2\mu_2} A^{\mu_2-1} \\ \vdots \\ m_{p\mu_p} \\ \vdots \\ m_{p\mu_p} A^{\mu_p-1} \end{bmatrix}$$

Using transfer matrix T , transferring of the matrix A and B are,

$$\bar{A} = T^{-1}AT, \bar{B} = TB$$

Using desired poles $\{\lambda_1, \lambda_2, \lambda_3, \dots, \lambda_n\}$, the transferred canonical form of the system is

$$\bar{A} - \bar{B}K = \begin{bmatrix} 0 & 1 & 0 & \cdots & 0 & 0 & 0 & 0 & \cdots & 0 & 0 & 0 & 0 & \cdots & 0 \\ 0 & 0 & 1 & \cdots & 0 & 0 & 0 & 0 & \cdots & 0 & 0 & 0 & 0 & \cdots & 0 \\ \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots \\ 0 & 0 & 0 & \cdots & 1 & 0 & 0 & 0 & \cdots & 0 & 0 & 0 & 0 & \cdots & 0 \\ -\alpha_1 & -\alpha_2 & -\alpha_3 & \cdots & -\alpha_{\mu_1} & 0 & 0 & 0 & \cdots & 0 & 0 & 0 & 0 & \cdots & 0 \\ 0 & 0 & 0 & \cdots & 0 & 0 & 1 & 0 & \cdots & 0 & 0 & 0 & 0 & \cdots & 0 \\ 0 & 0 & 0 & \cdots & 0 & 0 & 0 & 1 & \cdots & 0 & 0 & 0 & 0 & \cdots & 0 \\ \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots \\ 0 & 0 & 0 & \cdots & 0 & 0 & 0 & 0 & \cdots & 1 & 0 & 0 & 0 & \cdots & 0 \\ 0 & 0 & 0 & \cdots & 0 & -\beta_1 & -\beta_2 & -\beta_3 & \cdots & -\beta_{\mu_2} & 0 & 0 & 0 & \cdots & 0 \\ & & \vdots & & & & \vdots & & & & & \vdots & & & \\ 0 & 0 & 0 & \cdots & 0 & 0 & 0 & 0 & \cdots & 0 & 0 & 1 & 0 & \cdots & 0 \\ 0 & 0 & 0 & \cdots & 0 & 0 & 0 & 0 & \cdots & 0 & 0 & 0 & 1 & \cdots & 0 \\ \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots & \cdots \\ 0 & 0 & 0 & \cdots & 0 & 0 & 0 & 0 & \cdots & 0 & 0 & 0 & 0 & \cdots & 1 \\ 0 & 0 & 0 & \cdots & 0 & 0 & 0 & 0 & \cdots & 0 & -\gamma_1 & -\gamma_2 & -\gamma_3 & \cdots & -\gamma_{\mu_p} \end{bmatrix}$$

or

$$\bar{A} - \bar{B}K = \begin{bmatrix} 0 & 1 & 0 & \cdots & 0 \\ 0 & 0 & 1 & \cdots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & \cdots & 1 \\ -\alpha_0 & -\alpha_1 & -\alpha_2 & \cdots & -\alpha_{n-1} \end{bmatrix}$$

Solving above matrix equation we will get gain matrix K.

b) Calculation of gain matrix

Applying above method, for the desired pole 0.1, -0.1, 0.1i, -0.1i, the gain matrix is

$$K = \begin{bmatrix} 93.8377 & -24.8771 & 0 & 0 \\ -24.1188 & 24.3614 & 0 & 0 \end{bmatrix}$$

V. SIMULATION

In absence of the input forces, the angles and their velocities increase rapidly which make the system unstable (see figure 3).

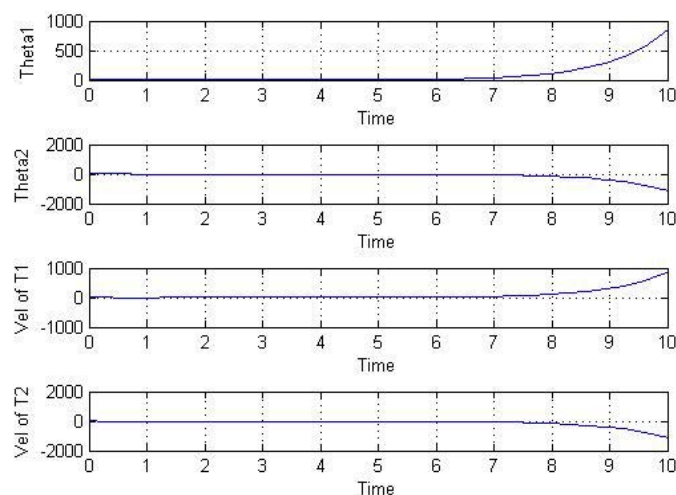


Figure 3: Uncontrolled System

By giving input force with measurement of gain matrix, the angles and their velocities will be slow down which make the system stable at the desired equilibrium place (see figure 4) .

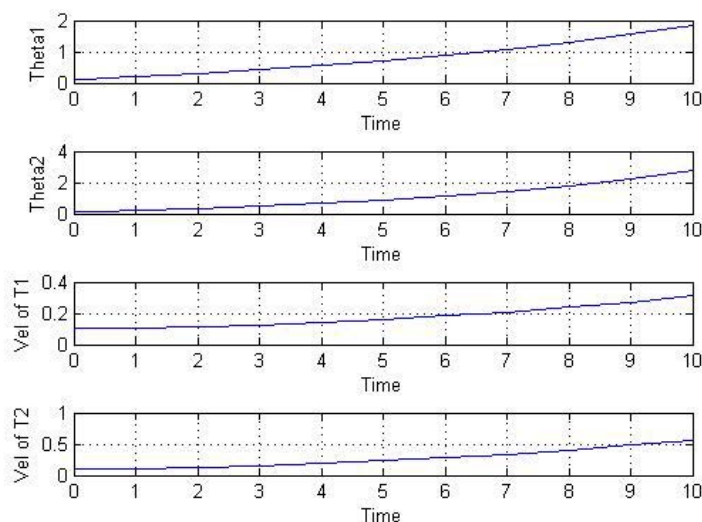


Figure 4 : Control System

VI. CONCLUSION

This study aims to understand what causes humanoids to fall, and what can be done to avoid it. Disturbances and modelling error are possible contributors to falling. For small disturbances in the walk of humanoid robot, it is simply behaving like a double inverted pendulum. So the results of this paper will be used in the development of the humanoid robot.

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