

Performance of a Capstone Gas Turbine based Power Plant Working on High Butane LPG

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Abstract

In this paper there are presented the results of the operational performance of a 30 kW microturbine generator (MTGs) fed with high butane content liquefied petroleum gas, while subjected to a stand-alone test procedure involving steady and transient load conditions. Neither modifications, nor regulations were made to the micro-turbine system for operation on the liquefied petroleum gas. To evaluate the performance, measurements of turbine and generator parameters were gathered from its original unit controller, as load changes were applied by changing load-bank values. For the stand-alone mode detailed graphs of the test results are presented, showing the transparency and robustness of the turbine-generator set to the used fuel, judging by the quality of the output electric parameters. The results from this performance testing provide good insight into the use of high-butane content liquefied petroleum gas as fuel for the tested microturbine. The continuous use of a fuel would need more tests to establish that the life of the critical components of the microturbine are not hampered from what they are on the baseline fuel.

Index terms— gas turbine, high butane LPG, electrical generator, performance, power generation.

1 I. Introducción

he need to respond in a safe, efficient and environmentally sustainable manner to the growing energy needs in the different sectors of the national economy, demands the rationalization, technical improvement and expansion of the sources of electricity supply. Responding to this demand, the Colombian Ministry of Mines and Energy, the Unit for the Mining-Energetic Planning (UPME), and the Energy and Gas Regulation Commission (CREG) lead a comprehensive policy that promotes, generates and stimulates programs and projects for the generation, saving and efficient use of energy and particularly for self-generation [1] [2] [4][5][6][7][8][9].

The Colombian Government issued Decree 2143 of 2015 [9], through which tax incentives are regulated for the promotion, development and efficient use of energy. The micro-grids find their way, with sources of distributed generation, local storage, controlled loads, and the possibility of developing electrical islands. Colombia is promoting programs for distributed generation (DG) and will probably encourage more projects for self-generation in the commercial, residential and service sectors, emulating initiatives as that of the US Department of Energy, which promoted the Advanced Alternative Engine Systems program (ARES), designed to develop small micro-generators units of high efficiency [12]. If incentives are created for self-generation in commercial, residential and service sectors, the incorporation into the system of microgeneration units could be attractive, and the introduction of microturbine generators could be favored.

On the other hand, Colombia is currently exporting LPG, a part of which is obtained as a byproduct of natural gas purification in known fields as Cusiana. Some energy suppliers have had interest in exploring the performance behavior of power generators when they are run on high butane liquefied petroleum gas for electricity generation in oil fields. Considering the fact that to date, to the authors' knowledge, there has not been reported any experimental tests related to the performance of microturbine generator (MTG) sets fuelled with LPG from Cusiana, a 30 kW Capstone MTG fed with Cusiana LPG was tested, as a pilot experience, to judge about its power output, step response, power quality, and fuel consumption.

4 FIGURE 3: C) EXPERIMENTAL FACILITY AND PROCEDURES

45 Microturbines are lightweight and compact in size combustion turbines with outputs of 30 kW to 400 kW that
46 can be used for stationary energy generation applications at sites with space limitations for power production.
47 They can be run on natural gas, biogas, propane, butane, diesel, and kerosene. Particularly, the Capstone
48 MTG consists of a compressor, recuperator, combustor, turbine and permanent magnet generator; the air drawn
49 through the inlet system refrigerates the generator, discarding the need of a liquid cooling system. Intake air
50 is compressed and injected into the recuperator, a heat exchanger where it is heated by turbine exhaust. Fuel
51 enters the system through an injection port and is mixed with the heated compressed air. The ignition system
52 causes the air-fuel mixture to burn in the combustion chamber under constant pressure conditions; the resulting
53 gases are allowed to expand through the turbine section to perform work, rotating the turbine blades to turn
54 a generator, which produces electricity. The rotating components, which T Global Journal of Researches in
55 Engineering (A) Volume Xx XII Issue I V ersion I can reach 96,000 min⁻¹ , are mounted on a single shaft
56 supported by low-maintenance air bearings.

57 The MTG has been tested in stand-alone mode, as a power source that meets the current consumption
58 demanded by the coupled load. The general goals of the load test were:

59 ? To get onsite experimental information related to the performance of the Capstone microturbine when fueled
60 with Cusiana LPG. ? To measure the electric generation performance of the MTG under a load cycle, for the
61 given open ambient conditions, with the available instrumentation, and the time allowed to perform the test,
62 adjusting as far as possible to the rules of operation and tests of the MTG.

63 ? To provide an appropriate stable medium for the reliable evaluation of electrical efficiency, and MTG
64 performance.

65 The work here presented refers to the evaluation study, and is organized as follows: first, the properties
66 of the LPG used are related, and a brief description of the Capstone micro-turbine is given. After that, this
67 paper describes the experimental procedure and constraints. Next, the test program is described, followed by a
68 summary of the results. Finally, the main conclusions of the work are presented.

69 2 II. Materials and Methods

70 3 a) Particularities of the LPG from Cusiana

71 The term LPG applies widely to any mixture of propane and butane, the two constituents occurring naturally
72 in oil and gas reservoirs that are gaseous at normal atmospheric conditions but can be liquefied by pressure
73 alone. Components heavier than butane are liquids at normal conditions and components lighter than propane
74 cannot be liquefied without refrigeration. The presence of butane, pentane, and heptane at concentrations of
75 up to 40% characterize this particular LPG from Cusiana, which analysis is presented in table 1. BTU?ft⁻³
76 @ 14,65psia, 60°F b) Capstone micro-turbine Microturbines have advantages over modern internal combustion
77 engines, such as their high-power density, less moving parts and comparatively low emissions. They can be fuelled
78 by liquid and gaseous fuels -fossil or renewable. Microturbine capacities are generally between 30 to 350 kW.
79 The Capstone 330 MTG, made available by the company Supernova Energy Services, installed in Alsabana was
80 subjected to service setting works, as it was new. Photographs in figure 1 allow to illustrate the general view of
81 the MTG located at the test site. Since Capstone Microturbines use lean premix combustion system to achieve
82 low emissions levels at a full power range, they require operating at high air-fuel ratio; injectors control the
83 air-fuel ratio. The MTG is instrumented to record operational parameters (of which, temperatures, pressures,
84 fuel usage, turbine speed, internal voltages/currents, and status are of importance for the undertaken study).
85 The average readings of two thermocouples indicates the Turbine Exit Temperature (TET); a compressor inlet
86 thermistor is installed to measure the air temperature at the inlet of the compressor wheel; the air flow, Wair,
87 (in pounds per hour) and the amount of energy needed in the combustion chamber required to regulate fuel
88 flow in the combustion chamber, W energy (in Btu/sec), are calculated based on engine speed. Such data are
89 available with a computer or modem connected to an RS-232 port on the microturbine. A schematic drawing of
90 the built-in instrumentation supported by the microturbine generator is presented in figure 2. A large on-board
91 battery pack is used to start the microturbine, and also to store energy when the microturbine decelerates to
92 produce less power. To meet output power requirements automatically, the system can be configured in Auto
93 Load mode. Auto Load ensures that the microturbine closes the output contactor to immediately produce the
94 required output power once minimum engine load speed is reached. The output speed-power characteristic of
95 the microturbine generator is reproduced in figure ??.

96 4 Figure 3: c) Experimental facility and procedures

97 LPG was supplied from an oil field to the test site by a tanker with a storage pressure ranged between 50 and 90
98 psia during the test; an intermediary damper tank was used to reduce pressure fluctuations due to consumption,
99 and a bypass was used in the LPG supply line with its respective regulating valves before the entry of gaseous
100 fuel to guarantee the pressure, as can be observed in the photographs of figure 4. The average ambient conditions
101 at the time of the test were: temperature close to 14°C, 80,2% relative humidity, and 0,726 atmospheric pressure.
102 The procedure followed to assess electrical, thermal, and operational performance of the microturbine generator,
103 comprised pretest activities, startup, idle, and a two-step load test during a short period of time. The software

104 for the microturbine unit was configured for standalone operation through the local display panel; the turbine
105 was started, controlled and monitored by a computer using Capstone's software.

106 The load test applied by the load bank, as it is shown in figure 6, consisted of a transient from idling to
107 a 20-kW load at maximum speed of 96000 min⁻¹, a steady-state operation in this operation point for about
108 eight minutes, followed by a drop to 3 kW power at 60000 min⁻¹, and a steady running at this load for about
109 four minutes. In the last part of the test, the load was completely released and the microturbine was sustained
110 idling at a speed of 45000 min⁻¹, as it is shown in figure 6. During the test, all available parameters were
111 monitored and recorded from the MTG system. All electrical parameters (both single-phase and three-phase)
112 were recorded at the load bank by the energy meter. The study focused on the overall performance parameters
113 related to engine operation. In the following, the results of the collected data during the operation cycle, and
114 the response of the turbine-generator to load changes are presented. Initially, the results obtained from the
115 proprietary MTG controller are presented: inlet to compressor and turbine exhaust gas temperatures, air flow,
116 intake air temperature and pressure values. Once the behavior variables are described, the evolution graphs of
117 the electrical power, voltage, and current delivered by the MTG are illustrated. The information thus presented
118 allows to evaluate the behavior of the MTG operating with LPG, from the perspective of stable operating capacity
119 and within the mechanical, thermal, environmental limits.

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121 6 a) Mean-variable measurement results

122 The microturbine generator has presented a normal behavior during the test, judging by the values of the speed,
123 power, inlet to the turbine and compressor temperatures, load percentage, among the operating parameters
124 registered by the proprietary controller of the microturbine; a summary of those performance parameters is
125 presented in table ???. The general behavior of the MTG during the test, as a function of time, is presented in
126 figure 7, where the history of output power, rotation speed, inlet to compressor and exit turbine temperatures,
127 input amount of energy, and air flow is plotted. Analysis of the graphs shows that the Capstone microturbine
128 responds to load changes rapidly, yet during steps up and down in the MTG real power output, turbine speed
129 follows ramps up and down smoothly to the new operating point. When the load bank resistance is reduced,
130 turbine shaft speed drops smoothly to its new operating point.

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132 (A) Volume Xx XII Issue I V ersion I Considering the heat value given by the chromatographic analysis of
133 the LPG, the energy flows are converted to fuel flows, which allows to approximate also the air/fuel ratio. The
134 variation of these magnitudes is shown in the figure 8.

135 8 IV. Results of the Electrical Energy Generated

136 The quality of the energy generated could be influenced by the quality of combustion process of the fuel. MTG
137 with the Capstone microturbine meets the specifications demanded for class G1 generators, in terms of frequency
138 and voltage deviations during transient processes. Observation of the voltage at the load banks during the test
139 showed a small sensitivity to load level. There is little change in the balance of the three-phase voltages. There
140 is no discernable pattern to the changes in the bus voltage; all three phases respond equally to the load changes.

141 9 V. Conclusions

142 Microturbine pilot test was carried out to determine performance characteristics, and to assess the possible
143 inconveniences of using the particular highbutane-content LPG. To achieve the technical goals, it was considered
144 a short test based on a two-step sequence of loads. The main conclusion drawn from the study is that, under the
145 scope of this study, the output of the turbine generator was satisfactory, showing its adaptability to the change
146 in fuel. The limited amount of testing done here restricts the applicability of these conclusions to the specific
147 type of LPG used. Cold and warm starts performed well. The estimated efficiency of power generation appeared
148 to be unchanged, as compared to the values indicated by the manufacturers.

149 It can be stated that, based on the short test carried out, gas turbines are an advantageous alternative to
150 the use of reciprocating engines, due to their adaptability to the fuel, their low noise and vibration levels, their
151 compact structure and their efficiency close to that of the diesel engine. The tests showed a higher-than-expected
152 performance and it is about to find out with the supplier how close this value is, since in the literature itself a
153 performance of more than 30% is not expected, while the conducted test showed an efficiency close to 40% for
154 66% of the load. It is yet to be proven.

155 Gas turbines are an excellent alternative and their technology is very mature for generation and cogeneration
156 standalone applications; in the commercial and industrial sectors, microgrid power parks, remote off-grid



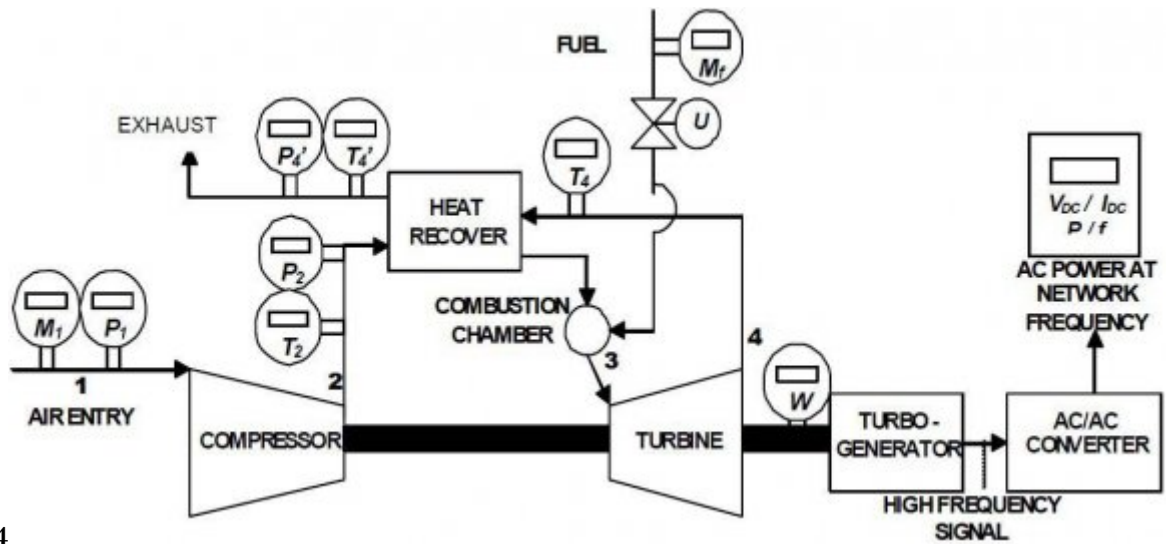
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Figure 1: Figure 1 :



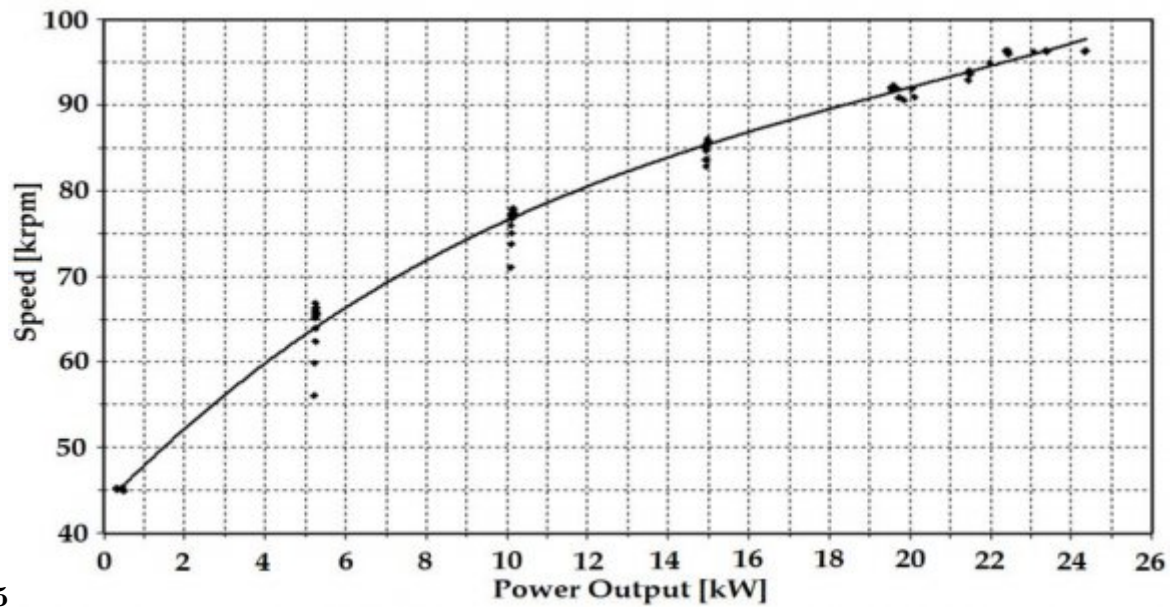
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Figure 2: Figure 2 :



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Figure 3: Figure 4 :



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Figure 4: Figure 5 :

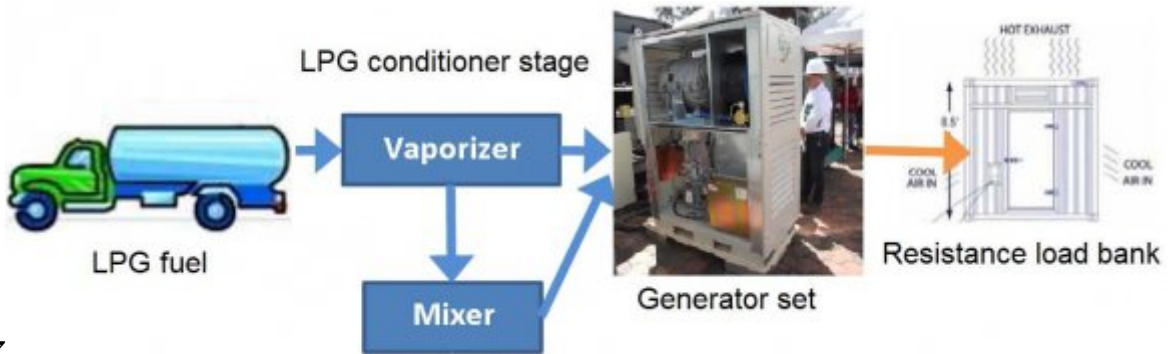


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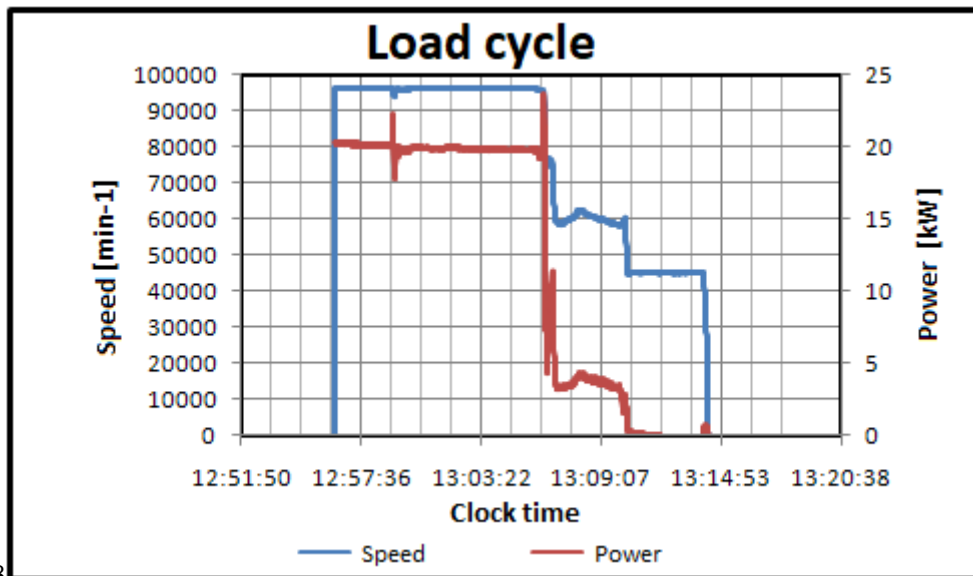
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Figure 6: Figure 6 :



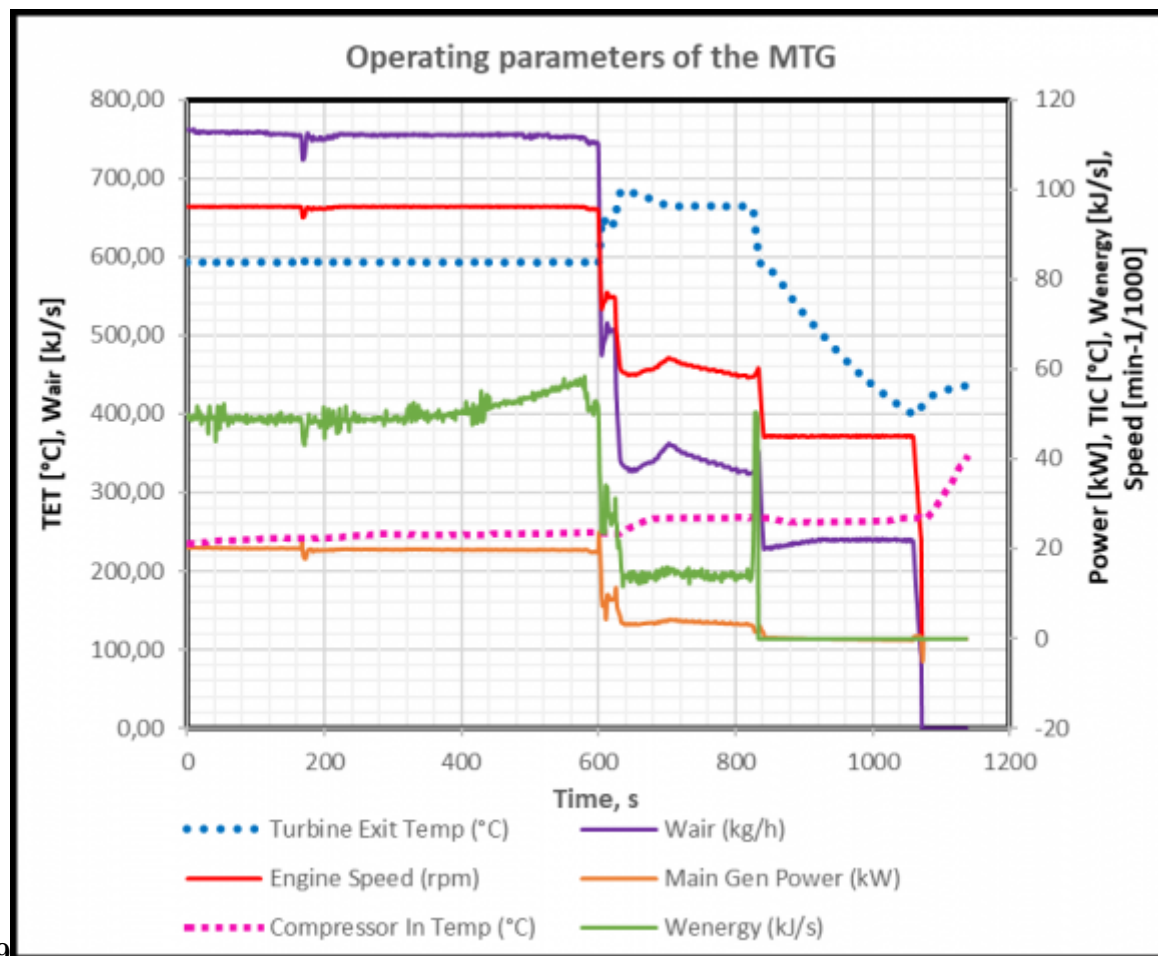
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Figure 7: Figure 7 :



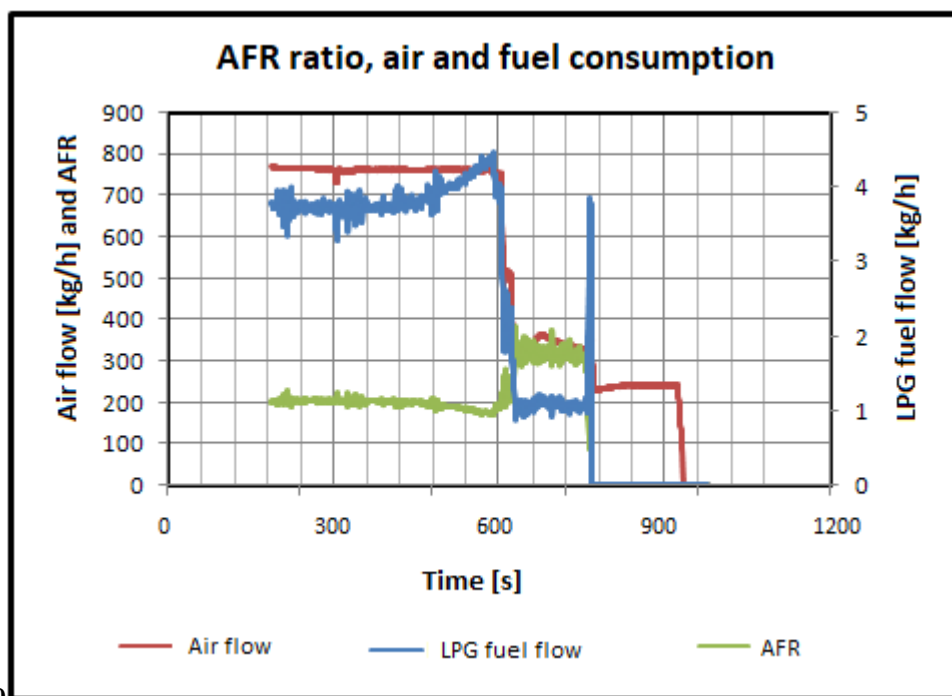
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Figure 8: Figure 8 :



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Figure 9: Figure 9 :



10

Figure 10: Figure 10 :

1

Compositional Analysis of GLP to C12+			
SamplingLocation	ALSABANA		
CylinderNumber	CLM009		
SamplingConditions	30,0 psig @ 66.0°F		
Component	Mole % Weight %		
CO2	CarbonDioxide	0,01	0,01
N2	Nitrogen	0,10	0,06
C1	Methane	0,01	0,00
C2	Ethane	3,36	2,14
C3	Propane	71,45	66,84
iC4	i-Butane	13,13	16,20
nC4	n-Butane	11,91	14,70
iC5	i-Pentane	0,03	0,05
Totals :		100,00	100,00
Note: 0,00 means less than 0,005.			
Calculated Whole Gas Properties			
Gas Gravity	1,6272	(Air = 1 @ 14,73 psia & 60°F)	
Whole Sample Mole Weight	47,13	g mol ⁻¹	
Ideal Gas Density	1,9831	kg m ⁻³ @ 14,65 psia, 60°F	
Ideal Gross Calorific Value	2665,7	BTU?ft ⁻³ @ 14,65 psia, 60°F	
Ideal Net Calorific Value	2454,5	BTU?ft ⁻³ @ 14,65 psia, 60°F	
Pseudo Critical Press.	598,7	psia	
Pseudo Critical Temp.	682,1	Rankine	
Gas Compressibility Factor, Z	0,979184	@ 14,65 psia & 60°F	
GPM (C2+)	28,49		
GPM (C3+)	27,60		
Additional Information			
Real Gross Calorific Value	2722,4	BTU?ft ⁻³ @ 14,65 psia, 60°F	
Real Net Calorific Value	2506,6		

Figure 11: Table 1 :

2

Figure 12: Table 2

9 V. CONCLUSIONS

2

Fuel type	GNC (55 psig)	GLP (55 psig)	Diesel (5 psig)
Mean time to repair	20,000 h	20,000 h	20,000 h
Nominal full power	30 kW net (+/-1 kW)	30 kW net (+/-1 kW)	29 kW net (+/-1 kW)
Peak efficiency (LHV**)	27% (+/-2%)	27% (+/-2%)	26% (+/-2%)
Fuel consumption***	18,7 lb/h, 8,5 kg/h	19,0 lb/h, 8,6 kg/h	21,9 lb/h, 10,0 kg/h
Methane based fuel flow (Metan-HHV)	440,000 kJ/h (420,000 Btu/h)		
Methane based energy of exhaust gases	305,000 kJ/h (290,000 Btu/hr)		
Exhaust gases temperature	500°F, 261°C	500°F, 261°C	500°F, 261°C
Output voltage	250-700 VDC	250-700 VDC	250-700 VDC

[Note: * Source: Capstone Turbine Corporation]

Figure 13: Table 2 :

7

Tiempo [min:s]	Speed [min -1]	Power [W]	TET [F]	TEC [F]	Wair [pph]	Amb. pressure [psia]	Suplied energy (btu/s) W energy	Accel. [%]	Frequency [Hz]
00:0	96320	20277	1102,2	69,9	1694	10,7	46,6	60,8	60
00:01	96236	20369	1101,2	69,9	1695	10,7	47,2	60,3	60
02:49	94028	18104	1109,8	72	1614	10,6	41	57,1	60
10:00	91236	23675	1110,1	74,4	1532	10,6	42,3	58,5	60
10:01	86408	21181	1129,9	74,4	1394	10,6	35	53	60
10:02	81252	17794	1144,6	74,4	1265	10,6	32,4	49,4	60
10:03	76490	14693	1164,9	74,4	1138	10,6	29,1	80,9	60
10:04	73480	9882	1203,5	74,2	1057	10,6	22,5	42,4	60
10:43	58912	3206	1259,8	75,9	737	10,7	12,8	35,7	60
10:45	58678	3432	1258,4	76	728	10,7	14	36,7	60
13:51	59852	1629	1126,8	80,4	772	10,7	46,6	100	60
13:55	54208	2411	1097,5	80,4	667	10,7	0	0	60
13:56	52114	2144	1095,2	80,2	630	10,7	0	0	60
13:57	50106	1885	1094,5	80,2	594	10,7	0	0	60
14:36	44950	120	1029,2	78,8	518	10,6	0	0	60

[Note: *]

Figure 14: Table 7 :

157 locations, presenting only the defect of greater starting time (close to 2 minutes, value obtained from the
158 literature). Engine Speed [min ??1] ^{1 2 3}

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-
- 159 [Capstone Turbine Corporation, Capstone Microturbine Model 330 System Operation Manual ()] *Capstone*
160 *Turbine Corporation, Capstone Microturbine Model 330 System Operation Manual*, 2000. 2001. USA.
161 Capstone Turbine Corporation, Capstone Low Emissions Microturbine Tecnology, White Paper, USA
- 162 [Capstone Turbine Corporation, Capstone Microturbine Product Catalog ()] *Capstone Turbine Corporation,*
163 *Capstone Microturbine Product Catalog*, <http://www.capstoneturbine.com/prodsol/products/>
164 2012. USA. p. .
- 165 [Capstone Turbine Corporation. 0911 C30 Natural Gas Data Sheet CAP135 | Capstone P/N 331031E]
166 http://www.capstoneturbine.com/docs/datasheets/C30%20NatGas_331031E_lowres.pdf
167 *Capstone Turbine Corporation. 0911 C30 Natural Gas Data Sheet CAP135 | Capstone P/N 331031E*,
- 168 [Consultancy to Determine the Schemes for Use of Derived Liquid Petroleum Gas Surplus for Electricity Generation in Oil Fields
169 *Consultancy to Determine the Schemes for Use of Derived Liquid Petroleum Gas Surplus for Electricity*
170 *Generation in Oil Fields*, April 18, 2013. Bogotá.
- 171 [Oyakawa ()] ‘Development of Ceramic Gas Turbines’. K Oyakawa . *JICA. Tsukuba* 1994.
- 172 [Rosa Do Nascimento ()] ‘Development of Ceramic Gas Turbines in Japan’. Rosa Do Nascimento . pg. 10-22.
173 Tokio. *Techno Japan* 2014. 1997. 30. Federal University of Itajubá, Brazil (Micro Gas Turbine Engine: A
174 Review)
- 175 [González et al. ()] ‘Estrategias de control de calidad de energía en microrredes rurales’. N Y González , C
176 Cusgüen , E Mojica-Nava , A Pavas . 10.18273/revuin.v16n2-2017009. [https://doi.org/10.18273/](https://doi.org/10.18273/revuin.v16n2-2017009)
177 [revuin.v16n2-2017009](https://doi.org/10.18273/revuin.v16n2-2017009) *UIS Ingenierías* 2017. 16 (2) p. .
- 178 [Wang et al. ()] ‘General characteristics of single shaft microturbine set at variable speed operation and its
179 optimization’. W Wang , R Cai , N Zhang . doi:101016/japplthermaleng200312012. *Applied Thermal*
180 *Engineering* 2004. 24 (13) p. .
- 181 [Liquefied petroleum gas code National Fire Protection Association. NFPA ()] ‘Liquefied petroleum gas code’.
182 *National Fire Protection Association. NFPA* 2011. 2011. 58.
- 183 [Bayar (2015)] *Microturbines take on the market. Cogeneration & on-site power production*, T Bayar . October
184 2015. p. .
- 185 [Ho et al. ()] *Performance study of a microturbine system for cogeneration application, Renewable Energy*, J C
186 Ho , K J Chua , S K Chou . 2004. p. .
- 187 [Nishiyama and Iwai M ()] ‘Status of the Automotive Ceramic Gas Turbine Development Program, Year Five
188 Progress. Artículo ASME 96-GT-36’. T Nishiyama , Iwai M . *ASME Journal* 1996.
- 189 [Nishiyama and Iwai M ()] ‘Status of the Automotive Ceramic Gas Turbine Development Program, Year Four
190 Progress. Artículo ASME 95-GT-447’. T Nishiyama , Iwai M . *ASME Journal* 1995.
- 191 [Itoh and Kimura ()] ‘Status of the Automotive Ceramic Gas Turbine Development Program. Artículo ASME
192 92-GT-2’. T Itoh , H Kimura . *ASME Journal* 1992.