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| 1 | Optimization of Diesel Engine Parameters for Performance, |
|---|---|
| 2 | Combustion and Emission Parameters using Taguchi and Grey |
| 3 | Relational Analysis |
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⁸ Abstract

19

Design and operating parameters of diesel engine were optimized in the present work with respect to performance, combustion and emission parameters. The goal is to reduce brake 10 specific fuel consumption (BSFC), exhaust gas temperature (EGT), ignition delay (ID), 11 emissions (CO, NOx, HC) and to increase peak pressure (PP), brake thermal efficiency 12 (BTHE), heat release (HR) simultaneously with least number of experimental runs. The 13 objective was accomplished through experimental investigations, design of experiments, 14 Taguchi method and Grey Relational Analysis. Four parameters viz. injection timing (IJT), 15 injection pressure (IP), compression ratio (CR) and load were varied at four levels and the 16 (nine) responses were recorded. Taguchi approach was applied to individual response and 17 observed that optimal factor settings for various responses are different. 18

Index terms — diesel engine, performance parameters, combustion parameters, emission parameters, taguchi
 approach, signal to noise ratio, grey relational approach.

²² 1 Optimization of Diesel Engine Parameters for Performance, ²³ Combustion and Emission Parameters using Taguchi and ²⁴ Grey Relational Analysis

Abstract-Design and operating parameters of diesel engine were optimized in the present work with respect to 25 performance, combustion and emission parameters. The goal is to reduce brake specific fuel consumption (BSFC), 26 exhaust gas temperature (EGT), ignition delay (ID), emissions (CO, NOx, HC) and to increase peak pressure 27 (PP), brake thermal efficiency (BTHE), heat release (HR) simultaneously with least number of experimental runs. 28 The objective was accomplished through experimental investigations, design of experiments, Taguchi method and 29 Grey Relational Analysis. Four parameters viz. injection timing (IJT), injection pressure (IP), compression ratio 30 (CR) and load were varied at four levels and the (nine) responses were recorded. Taguchi approach was applied to 31 32 individual response and observed that optimal factor settings for various responses are different. Grey relational 33 approach (by assigning weighting factor for each response) was applied to solve multi objective optimization 34 problem. The optimal combination of factors was obtained as injection timing 28 0 bTDC, injection pressure 35 180 bar, compression ratio 19 and load 80% full load and load was observed to be most influential factor among the four with a contribution of 70.37%. The model developed was validated by confirmation test and found good 36 agreement between predicted and experimental values of responses. 37 here is a huge demand for diesel engines in industrial, agricultural and automotive sector. The advantages of 38

diesel engines over gasoline engines are fuel economy, high thermal efficiency, low CO 2 emissions, ruggedness, flexibility to operate at higher compression ratio and so on. However, faster depletion of fossil fuels and environment pollution demand the engine designers to take control over fuel economy and emissions. Much research has been carried out to tackle these problems. Various investigators attempted to optimize engine design and/or operating parameters with respect to performance, combustion and emission parameters to control fuel economy or emissions. Diverse numerical and statistical techniques are available for optimization; however, some offer single objective optimization and some other multi objective. The benefit of multi objective or multivariate optimization over single objective optimization is that influence of factors on multiple responses can be assessed

47 and studied.

48 2 a) Design of experiments

Most processes depend on some controllable factors. Similarly, performance, combustion process and emissions of 49 a diesel engine, depends on design parameters like injection timing, injection pressure, compression ratio, engine 50 size, type of combustion chamber and operating parameters such as load, intake temperature and pressure of air, 51 speed, air-fuel ratio etc. To realize the effect of control factors on responses like; performance, combustion and 52 emission parameters, a series of experiments are to be run. Experiments are to be well designed for generating 53 more significant information within fewer runs to evaluate the important effects, rather than employing unplanned 54 55 experiments. Design of Experiments (DOE) offer systematic investigation of the control factors that influence the 56 responses. Common methods in DOE are hit and miss, one factor at a time, full factorial, fractional factorial etc. Fractional factorial method has advantage of less number of experiments without loss of much information and 57 58 adopted in the present work. The four phases involved in DOE are planning, screening or process characterization, 59 optimization and verification. Planning includes defining the problem and objective, followed by development of experimental plan which provides all significant information. Screening or process characterization comprises 60 of selection of factors which are really important, or the vital few. Various methods are available for screening 61 but most widely used is fractional factorial method. In fractional factorial method, only a selected subset or 62 fraction of runs in the full factorial design is performed. Optimization is the phase where best or optimal values 63 of control factors are determined by various techniques available. Verification is the phase in which optimal factor 64 65 values after prediction are tested for confirmation of results. In planning phase, factors, levels and responses are 66 chosen and orthogonal arrays are formed, which were originally developed by Sir R.A. Fisher and later added by Taguchi. Experiments are conducted based on orthogonal arrays and analysed based on Signal to Noise Ratio 67 68 (SNR). Signal to Noise Ratio (SNR) measures the variation of response relative to nominal or target value. Two types of SNR used in the present work; smaller the better and larger the better. 69

Numerous studies have been carried out to study the effect of IJT, IP, CR and load of the engine on performance, combustion and emission parameters [1][2][3][4][5]. For better performance, the engine should be operated at a set of optimal design and operating parameters. Optimization with the help of orthogonal arrays was proposed by Taguchi [6] in which optimum set of factors is determined for each response with the help of (Signal to Noise Ratio) SNR. Taguchi method has been applied successfully for numerous problems in various fields of science and technology. Diesel engine parameters are not an exception for it. A brief review of research carried is following.

77 T. Ganapathy et al. [7] used Taguchi method to optimize ten operating and design variables of diesel engine 78 for maximum brake thermal efficiency, peak pressure, temperature, IMEP, BMEP and reported improvement in above said parameters at optimal condition obtained by Taguchi approach. Horng-Wen Wu et al. [8] reported 79 that Taguchi method is good to find optimal operating parameters for high brake thermal efficiency and low 80 BSFC, NO x and smoke. Kaliamoorthy.S et al. [9] employed Taguchi method to optimize power, static injection 81 timing, fuel fraction and compression ratio for best values of brake power, fuel economy and emissions and 82 reported that confirmation tests showed good agreement with predicted values of parameters. Karthikeyan. R 83 84 et al.

[10] from their work concluded that Taguchi method of optimization efficiently predicted optimum level of parameters and found satisfactory results at optimum setting. The inference from the work of Vincent H. Wilson et al. [11] confirmed that Taguchi method is efficient in predicting range of optimum settings of valve opening pressure, piston to head clearance volume, static injection timing, area of the spray nozzle hole and load for best values of NO x emissions and brake specific fuel consumption.

Even though Taguchi method proved as one of the best methods for optimization, its major limitation is 90 inability to tackle multi objective optimization. This drawback is trounced by application of grey relation analysis 91 and Taguchi method collectively. Grey relational analysis, proposed by Deng in 1982, which is commonly used 92 for assessing the degree of correlation between sequences by grey relational grade. In this analysis, responses 93 are normalized (between zeros to one) which is known as grey relational generation. Grey relational coefficient 94 95 is calculated using normalized data of responses. Grey relational coefficients of all the responses is averaged to 96 get overall grey relational grade. The calculation of grey relational grade converts multi variant optimization 97 problem into single response optimization, overall grey relational grade being objective function. By maximizing 98 the overall grey relational grade the optimal parametric combination is evaluated. Some research work is also reported regarding use of grey relational analysis in conjunction with Taguchi method. 99 The results of research done by Ashish Karnwal et al. [12] emphasized that Taguchi method coupled with grey 100

relational analysis can be used successfully for exploration of multiple-performance variables of diesel engine. In their work, biodiesel blend, compression ratio, opening pressure of nozzle and injection timing are optimized for best values of brake thermal efficiency, brake specific energy consumption and exhaust gas temperature of diesel engine. In a study carried out by Sumit Roy et al. [13] optimization of CNG energy share and fuel injection
 pressure for lowest values of BSFC, NO x and HC done successfully.

Goutam Pohit et al. [14] reported effective optimization of biodiesel blend, compression ratio and load for 106 better values of performance and emission parameters by grey relational analysis and supported by confirmatory 107 experiments. Optimization of speed of the engine, load and type of fuel for better values of performance and 108 emissions was prolifically done by M. I. Masood et al. [15] by means of grey Taguchi method and confirmatory 109 test by artificial neural networks showed best validation. Taguchi method along with grey relational analysis and 110 ANOVA was able to identify the order of significance/ contribution of each of the parameters (injector opening 111 pressure, fuel injection timing and compression ratio) on BTHE, BSFC and emissions, further they reported that 112 113 (1) n S i i SNR y n = ? ? ? ? = ? ? ? ? ? ? ? ? ? Optimization of 114

115 **3** A

Where y i is the response from i th experiment and i=1, 2 ...n. After SNRs are evaluated, main effect plot for SNRs are drawn to find optimal values of the factors. ANOVA is performed to explore and model relationship between responses and factors and relative percentage contribution of factors on response.

When the response is to be minimized, 'smaller the better' SNR is appropriate and is computed using the Eq. 1 while 'larger the Better' SNR is apt for the maximizing response applying the Eq. 2.

results were in good agreement with predicted values [16]. Similar results were also reported by some other

¹²² 4 b) Motivation and Objectives

As per available literature, most of the research work pertaining to diesel engine parameters was concentrated on either of performance or emission parameters or both. However, so far no work was reported on optimization for parameters of combustion like peak pressure, ignition delay, and heat release together with performance and emission parameters. Hence, objective of present work is to spot out optimal values of design and operating parameters of diesel engine, which would maximize brake thermal efficiency, peak pressure, and heat release and to minimize BSFC, exhaust gas temperature, ignition delay and emissions simultaneously.

In the present work Taguchi method and grey relational analysis are used for optimization. Taguchi analysis results shows order of factors influencing particular response in the form of ranks. Hence, ANOVA (Analysis of variance) is used to find the percentage contributions of IJT, IP, CR and load on response parameters.

¹³² 5 II. Materials and Methods

¹³³ 6 a) Experimental setup

Experiments were carried out on a 4-s singlecylinder, water cooled direct injection, variable compression ratio 134 diesel engine. The specifications of the engine are presented in Table1.and layout of engine in Fig. 1. The engine 135 is attached to an eddy current dynamometer with speed sensing unit incorporated. PCB (USA) make piezo 136 -electric transducer is flush mounted in the cylinder head and used to measure cylinder pressure. An optical 137 encoder is employed to capture the rpm of the crank shaft. Data acquisition system with high speed is used 138 for acquisition and analysis of pressure crank angle data is done by software. To eliminate effect of cycle to 139 cycle variation, pressure crank angle data for 100 consecutive cycles is recorded and averaged. Parameters are 140 calculated using averaged data. Software calculates and displays performance and combustion parameters from 141 the recorded observations. Each experiment was conducted four times and values are averaged to avoid errors. 142 Emissions are measured by a gas analyzer; specifications are presented in Table 2. 143

investigators ??17, 18.

¹⁴⁵ 7 III. Methodology

The factors considered in the present work are injection timing, injection pressure, compression ratio and load. 146 Each factor is varied at 4-levels as presented in Table 3. The Taguchi method employs orthogonal arrays from 147 theory of DOE to learn the effect of huge number of controllable factors on responses inside a small experimental 148 matrix. Use of orthogonal arrays notably reduce the number of experiments in view of the fact that it provides 149 the shortest possible matrix in which all factors are varied over working range. Furthermore, the conclusions 150 151 from this shortest number of experiments are valid over entire range. L 16 orthogonal array is prepared from 152 Taguchi's design with four factors and four levels as presented in Table 4. Interaction among the factors was 153 neglected because all are independent. Motivation for selection of response variables is, to make the present work 154 significant to the existing studies of focussing all pervading performance, combustion and emission parameters that confront the contemporary diesel engine design. 155

Various performance parameters considered in the present work are brake specific fuel consumption (BSFC), brake thermal efficiency (BTHE) and exhaust gas temperature (EGT). Low values of BSFC and EGT are preferable whereas high value of BTHE is preferable. BSFC and BTHE were calculated and EGT was recorded

159 from display.

Combustion parameters studied in the present work are ignition delay (ID), peak pressure (PP) and Heat 160 release (HR). The time interval between start of injection (CAD at which fuel injection starts) and start of 161 combustion (CAD at which combustion starts) in diesel engines is called ignition delay period [19]. It may be 162 expressed in terms of CAD (crank angle degrees) or milliseconds. From p-? data, CAD is noted, where positive 163 values of heat release (start of combustion) is observed and ignition delay was calculated as difference between 164 CAD of start of combustion and start of injection. 165

Peak pressure is the maximum cylinder pressure attained during combustion process very near to and after 166 TDC and is taken from p? data. Heat release is the amount of heat released during combustion process. 167 According to Heywood [19], combustion continues well into the expansion stroke up to 31 o. HR is taken as the 168 sum total of HR per CAD from start of combustion to significant positive values of HR per CAD. 169

NO x , CO and HC are the emissions considered in this work for analysis and are measured using exhaust gas 170 analyser (details are presented in Table ??2) 171

8 a) Taguchi Method 172

Taguchi approach employs the parameter SNR (Signal to Noise Ratio) for optimization. Largest value SNR 173 is preferred as it indicates minimized effects of noise factors. SNRs are calculated by formulae mentioned in 174 section 1.2 based on criteria smaller the better or larger the better. Larger the better criteria is used for brake 175 thermal efficiency, peak pressure, and heat release whereas smaller the better criteria is used for BSFC, exhaust 176 gas temperature, ignition delay, CO, NO X and HC. In the present work Minitab software is used for Taguchi 177 design, SNR calculations, main effects plots and performing ANOVA. After computation of SNRs, main effect 178 plots for SN Ratios are plotted by taking data means. The SNRs for different responses were calculated at each 179 factor level. The average effects were calculated by taking sum total of each factor level and then dividing by 180 number of data points. 181

In view of the fact that Taguchi approach results in different optimal conditions for various responses, overall 182 optimal condition cannot be figured out. Hence in the present work grey relational analysis is also carried out 183 for multi objective optimization. 184

9 b) Grey Relational Analysis 185

The degree of approximation among the sequences is measured using a parameter called grey relational grade 186 in grey relational analysis. In grey relational analysis, the responses are normalized between zero and 1. This 187 process is known as grey relational generation. Normalized data for lower the better criteria can be calculated 188 by Eq.3 and for higher the better by Eq. 4. 189

max () () () max () min () 10 190

i i i i i y k y k x k y k y k ? = ?(3)191

() max () () max () min () i i i i i y k y k x k y k y k ? = ?(4) 192

for comparison (normalized value of response) and i=1,2,...,m and k=1,2,...,n; m is total number of 193 experiments and n is total number of responses. min y i (k) and max y i (k) are lowest and highest values of 194 y i (k)respectively. Next, deviational sequences \hat{I} ?" of for responses are calculated from Eq.5. 195

GRC (Grey relational coefficient)? i (k) for each response is calculated to represent the correlation between 196 the desired responses and actual experimental data using Eq. 6 .min max max () () i oi k k ???? +? =? 197 +?(6)198

Î?" min and Î?" max are the minimum and maximum values of the absolute differences of all comparing 199 sequences. ? is the distinguishing coefficient and it lies in the range 0 ? ? ? 1. Value of distinguishing coefficient 200 is taken as 0.5 for all responses [20,21]. Experiments were conducted as per L 16 orthogonal array presented in 201 Table 4. Compression ratio was varied with the help of a lever attached to the cylinder head. Number of shims 202 was adjusted under the seat of the mounting flange of fuel pump to alter static injection timing. It was noted 203 that, addition of shims retards fuel injection timing and vice versa. Injection pressure was measured and adjusted 204 using an injector opening pressure test rig. It comprises of a pipe to connect to the injector and a fuel reservoir. 205 Spring tension of the nozzle is varied by adjusting screw on the injector to vary the pressure. 206

Global 11 207

Where y i (k) is the original sequence (response from experiments), x i (k) is the sequence where x 0 (k) was an 208 ideal sequence. 209

12210 oi i x k x k ? = ?(5)

211

13212

Subsequent to calculation of grey relational coefficients grey relational grade ? k is calculated for each response 213 by assigning appropriate weighting factor ? i. Weighting factor is assigned to a particular response, based on 214

their relative significance, and the sum of weighting factors must be equal to unity [22]. In the present work 215 weighting factor 0.2 is assigned for brake thermal efficiency and 0.1 for all other responses. 216

14() 217

n k i i k k ? ? ? = = ?(7)218

This study uses L 16 orthogonal array of Taguchi method mentioned in Table 4 to find out best Injection 219 timing, injection pressure, compression ratio and load setting for diesel engine. At four levels of each factor, the 220 responses viz. BSFC, brake thermal efficiency, peak pressure, and heat release, exhaust gas temperature, ignition 221 delay, CO, NO X and HC are determined. 222

IV. Taguchi Results Analysis and Confirmation Experiments 15223

SNR curves are graphical representations of variation in responses with variation in factor levels. From these 224 curves two observations are noted. First one is most influential parameters and second is their optimum levels. 225 After taking average of SNRs at four levels of particular factor, plots are drawn for means of SNRs Vs factor 226 level. Fig. 2 (a) to 2(i) show such plots for all 9 response variables. Level with highest value of mean SNR is 227 considered as optimal value. 228

16 Α 229

For k th response variable, where ? k is grey relational grade, ? i (k) is distinguishing coefficient, ? i is weighting 230 factor. Closeness of particular response with optimal value is given by higher value of grey relational grade. 231

A grey relational grade is a weighted sum of the grey relational coefficients, and is calculated using Eq. 7. 232

Experiments were conducted at optimal set of conditions as mentioned in Table 4 and corresponding responses 233 were recorded. The values of responses at optimal settings from Taguchi analysis are compared with that of 234

baseline engine and presented in Table 6. It is observed from Table 6 that the parameters BSFC, EGT, ID, CO, 235

NOx and HC shown significant decrease for optimized engine compared to baseline engine and is represented in 236

Fig. 3(a), whereas the parameters BTHE, PP and HR shown significant increase for optimized engine compared 237 238 to baseline engine and is represented in Fig. 3(b).

Fig. 2 (c) : Main effects plot for SNR of EGT 17239

$\mathbf{18}$ a) Grey Relational Analysis Results 240

Taguchi approach, even though resulted in optimal values of responses, factors are optimized one at a time 241 (single objective optimization) and for various responses different factor settings were obtained. To overcome 242 this problem with Taguchi approach, grey relational analysis with Taguchi approach was carried out for multi 243 objective optimization. 244

Initially responses were normalized based on higher the better or smaller the better criteria and deviation 245 sequences were calculated. Grey relational coefficients are calculated using Eq. 6 and grey relational grades for 246 responses were calculated using Eq. 7 by assigning appropriate weights and are presented in Table ??. The grey 247 248 relational grade signifies the correlation between the reference sequence and comparability sequence, higher value of grey relational coefficient indicates stronger correlation. From Table 9. it is concluded that optimal factor 249 setting is 4 th level of IJT i.e. 28 o bTDC, 1 st level of IP (180 bar), 4 th level of both CR and % full load i.e. 250 19 and 80% full load respectively. 251

To validate the model developed for optimize factor settings for maximum value of GRG prediction was 252 carried at IJT 24 0 bTDC, IP 200 bar, CR 16.5 and at 40% full load. GRG for prediction was 0.556263 where 253 as confirmation test by experimentation at the above factor settings was 0.551443. Further, from Table 9. it 254 is reported that most influencing factor is % full load whereas least one is injection timing. However, relative 255 importance of factors on responses quantitatively must be known for accurate determination of optimal factor 256 setting, which can be accomplished by ANOVA. 257

b) Analysis of Variance 19 258

The objective of Analysis of Variance (ANOVA) is to explore most influential parameter (factor) that effect 259 response, quantitatively. ANOVA is carried out using MINITAB software and results are presented in Table 10. 260

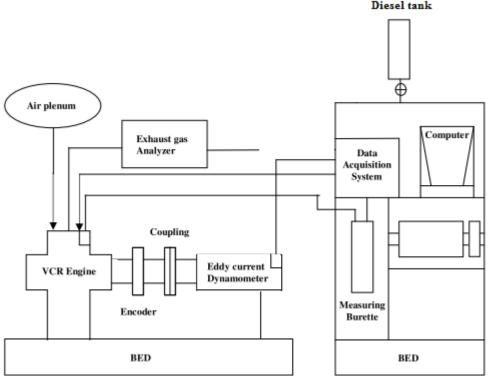
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Figure 1: Fig. 1 :



Diesel tank

Figure 2:

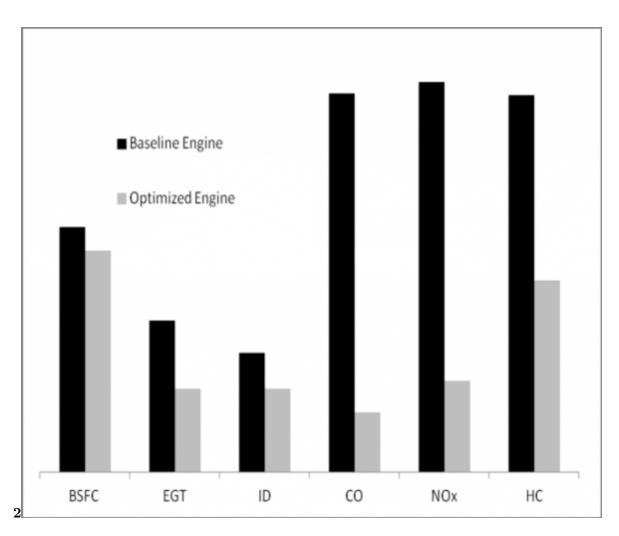


Figure 3: Fig. 2 (

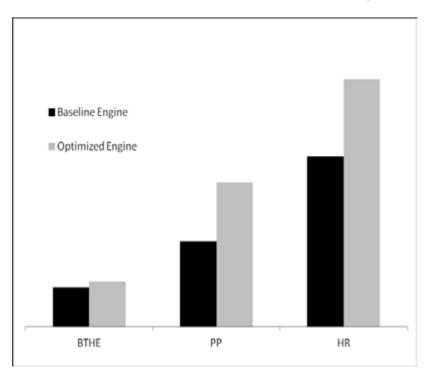


Figure 4:

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Figure 5:

1

| Engine Type | Kirloskar |
|--------------------------------|---------------------|
| Number of cylinders | Single(01) |
| Combustion | Direct injection |
| Bore | 80 mm |
| Stroke | $110 \mathrm{mm}$ |
| Compression Ratio | Variable $(15-20)$ |
| Rated Speed | $1500 \mathrm{rpm}$ |
| Power | 5 hp |
| Type of cooling | Water cooling |
| Fuel injector opening pressure | 200 bar |
| Fuel injection timing | 22 o before TDC |
| Type of loading | Electrical loading |

Figure 6: Table 1 :

$\mathbf{2}$

| Exhaust | Measurement | Resolution | Accuracy | Measuring |
|----------------------|------------------------|------------|-------------|-----------------|
| gas | | | | |
| | Range | | | Method |
| CO | 0-15.0% vol | 0.01%vol | + 0.06% vol | NDIR |
| HC | 0-30000 ppm (Propane) | 1 ppm vol | + 12ppm | NDIR |
| | 0-15000 ppm (Hexane) | | | |
| NO X | 0-5000 ppm | 1 ppm vol | + 50% vol | Electrochemical |

Figure 7: Table 2 :

3

| Factors | Level 1 Level 2 Level 3 Level 4 | | | | | |
|---------------------------|---------------------------------|------|-----|-----|--|--|
| Injection Timing (0 bTDC) | 20 | 22 | 24 | 26 | | |
| Injection Pressure (bar) | 180 | 200 | 220 | 240 | | |
| Compression Ratio | 15 | 16.5 | 18 | 19 | | |
| % of Full Load | 22 | 40 | 60 | 80 | | |

Figure 8: Table 3 :

 $\mathbf{4}$

| S.No | Injection | Injection | Compression | % |
|------|--------------------|-----------|-------------|------|
| | Timing(o bTDC) | Pressure | Ratio | Full |
| | / | (bar) | | Load |
| 1. | 22 | 180 | 15 | 20 |
| 2. | 22 | 200 | 16.5 | 40 |
| 3. | 22 | 220 | 18 | 60 |
| 4. | 22 | 240 | 19 | 80 |
| 5. | 24 | 180 | 16.5 | 60 |
| 6. | 24 | 200 | 15 | 80 |
| 7. | 24 | 220 | 19 | 20 |
| 8. | 24 | 240 | 18 | 40 |
| 9. | 26 | 180 | 18 | 80 |
| 10. | 26 | 200 | 19 | 60 |
| 11. | 26 | 220 | 15 | 40 |
| 12. | 26 | 240 | 16.5 | 20 |
| 13. | 28 | 180 | 19 | 40 |
| 14. | 28 | 200 | 18 | 20 |
| 15. | 28 | 220 | 16.5 | 80 |
| 16. | 28 | 240 | 15 | 60 |

Figure 9: Table 4 :

 $\mathbf{5}$

Figure 10: Table 5

 $\mathbf{5}$

| Controlled Factors | BSFC (kg/kW- | $\begin{array}{c} \text{BTHE} \\ (\%) \end{array}$ | EGT (0C) | $\begin{array}{c} \operatorname{PP} \\ (\operatorname{bar}) \end{array}$ | ID (CAD) | $\begin{array}{c} \mathrm{HR} \\ \mathrm{(J)} \end{array}$ | $\begin{array}{c} \mathrm{CO} \\ (\% \mathrm{by} \end{array}$ | $\begin{array}{c} \mathrm{NOx} \\ \mathrm{(ppm)} \end{array}$ | $\begin{array}{c} \mathrm{HC} \\ \mathrm{(ppm)} \end{array}$ |
|--------------------|-----------------|--|-------------|--|-------------|--|--|---|--|
| | hr) | | | | | | vol) | | |
| IJT (degrees BTDC) | 26 | 26 | 28 | 28 | 24 | 28 | 28 | 22 | 22 |
| IP (bar) | 200 | 200 | 180 | 240 | 240 | 240 | 240 | 180 | 180 |
| CR | 16.5 | 16.5 | 19 | 19 | 19 | 19 | 19 | 15 | 15 |
| % of Full Load | 80 | 80 | 40 | 80 | 80 | 80 | 40 | 20 | 20 |

Figure 11: Table 5 :

| 6 | | | | | | | | | |
|---------------------|------------------------|-------------|-------------|-------------|------------|-------------|------------------|--------------|-------------|
| | BSFC (kg/kW- hr) | BTHE (%) | EGT (0C) | PP (bar) | ID (CAD | HR) (J) | CO (% by vol) | NOx (ppm) | HC (ppm) |
| Baseline Engine | 0.41 | 18.95 | 253 | 40.9 | 20 | 817.74 | 0.0632 | 651 | 62 |
| Optimized Engine | 0.37 | 21.63 | 139 | 69.4 | 14 | 1185.68 | 0.0101 | 153 | 32 |

Figure 12: Table 6 :

| S.No Response Variable | | Taguchi | Confirmation | % Difference between pre- |
|------------------------|---------------------------|------------------|--------------|----------------------------------|
| | | Prediction value | test value | diction and confirmation test |
| | | Frediction value | test value | commination test |
| 1 | BSFC (kg/kW-hr) | 0.455688 | 0.42 | -8.49714 |
| 2 | BTHE $(\%)$ | 15.26 | 16.2 | 5.802469 |
| 3 | EGT (0 C) | 229.75 | 228 | 0.7617 |
| 4 | PP (bar) | 45.5812 | 44.4 | -2.66036 |
| 5 | ID (CAD) | 23.43 | 22.3 | -5.06726 |
| 6 | $\mathrm{HR}(\mathrm{J})$ | 606.519 | 623.6 | 2.739096 |
| $\overline{7}$ | CO (% by vol) | 0.05948 | 0.0622 | 4.37299 |
| 8 | NO (ppm) | 653.875 | 596 | -9.71057 |
| 9 | HC(ppm) | 58.875 | 55 | -7.04545 |

Figure 13:

| 77 | 8 |
|----|---|
|----|---|

| Wei | gûts | 0.2 | 0.1 | 0.1 | | 0.1 | 0.1 | 0.10.1 |
|----------------|---|--------|---------------|----------|--------|---------|---------------------|----------------|
| S.Ne | DBSFC BTHE | | EGT | PP | | ID | HR | CONOX |
| 1 | $0.3349\ 0.3335\ 0.5812\ 0.3333\ 0.3333\ 0.3333\ 0.33333\ 0.3862$ | 1 | | | | | | |
| 2 | $0.6266\ 0.4428\ 0.4583\ 0.4667\ 0.4366\ 0.39108\ 0.5544$ | 0.631 | $6\ 0.6986$ | 0.514963 | | | | |
| 3 | $0.9098 \ 0.7494 \ 0.3952 \ 0.5965 \ 0.6889 \ 0.49505 \ 0.6005$ | 0.414 | $2\ 0.4857$ | 0.608461 | | | | |
| 4 | $0.8868 \ 0.7146 \ 0.3333 \ 0.905$ | | | | | 1 | 0.63 | $613 \ 0.4766$ |
| 5 | 0.986 | 0.97 | 701 0.616 | 0.5022 | 0.5636 | 0.43081 | 0.39 | $051 \ 0.5825$ |
| 6 | 1 | 1 | 0.52 | 0.4555 | 0.5254 | 0.44138 | 0.33 | $333 \ 0.4816$ |
| $\overline{7}$ | $0.3333\ 0.3333\ 0.4617\ 0.6387\ 0.6078\ 0.52241\ 0.7994$ | 0.402 | $9\ 0.4016$ | 0.483446 | | | | |
| 8 | $0.5402 \ 0.4008 \ 0.3965 \ 0.6765 \ 0.6889 \ 0.57478 \ 0.8957$ | 0.338 | $9\ 0.3778$ | 0.529082 | | | | |
| 9 | $0.9591 \ 0.8628 \ 0.6552 \ 0.7708 \ 0.5254 \ 0.53367 \ 0.3868$ | 0.430' | $7 \ 0.4359$ | 0.642322 | | | | |
| 10 | $0.9098 \ 0.7559 \ 0.5812 \ 1$ | | | | | 0.563 | $6 \ 0.5$ | $7663 \ 0.575$ |
| 11 | $0.7622\ 0.5481\ 0.5576\ 0.3773\ 0.4026\ 0.49044\ 0.6336$ | 0.582 | $5\ 0.4766$ | 0.537909 | | | | |
| 12 | $0.4879\ 0.3803\ 0.4634\ 0.4148\ 0.4247\ 0.53775\ 0.6322$ | 0.511 | $9\ 0.4359$ | 0.466896 | | | | |
| 13 | $0.6812 \ 0.479$ | | 1 | 0.8956 | 0.4247 | 0.52378 | 8 1 | 0.4448 |
| 14 | $0.4983\ 0.3841\ 0.7792\ 0.7377\ 0.4133\ 0.51787\ 0.6927$ | 0.5342 | $2\ 0.4286$ | 0.536988 | | | | |
| 15 | 0.945 | 0.83 | $381 \ 0.601$ | 0.8147 | 0.4627 | 1 | | 0.398354 |
| 16 | $0.7231 \ 0.5095 \ 0.52$ | | | 0.5651 | 0.4493 | 0.75654 | 0.60 | $005 \ 0.3998$ |
| | | | | | | | | |

Figure 14: Table 7 Table 7 : Table 8 :

9

| LEVELS | IJT |
|--------|----------------------------------|
| 1 | $0.563433 \ 0.607479 \ 0.549412$ |
| 2 | $0.57363 \ 0.581263 \ 0.575372$ |
| 3 | $0.573982 \ 0.572939 \ 0.579213$ |
| 4 | $0.592598 \ 0.541962 \ 0.599646$ |
| Delta | $0.029165 \ 0.065517 \ 0.050235$ |
| Rank | 4 |

IP CR % FULL LOAD 0.496066 0.55373 0.613363 0.640484 0.144418 2 3 1

Figure 15: Table 9 :

The average of grey relational grade for each level of factor is calculated and tabulated in Table ??. From ANOVA results it is reported that load is the most influential parameter (70.37%) where as injection pressure, compression ratio and injection timing influence in the order is 12.37%, 7.22% and 2.51%.

²⁶⁷ .1 V. Conclusions

In this paper, optimal engine design and operating parameters viz. injection timing, injection pressure, compression ratio and % full load were determined for (nine) multiple response parameters (brake thermal efficiency, brake specific fuel consumption, exhaust gas temperature, peak pressure, ignition delay, heat release, CO, NOx and HC) by using Taguchi and grey relational analysis. 16 experiments were conducted as per L 16 orthogonal array.

As Taguchi approach can handle single objective optimization problem optimal factor settings for each of nine 273 parameters was explored separately, however it was observed that for various response parameters optimal factor 274 settings were different. Hence authors attempted multi objective/variant optimization by using grey relational 275 approach coupled with Taguchi approach. The grey relational analysis by assigning weighting factors, converts 276 optimization of multi response problem into optimization of single objective i.e. grey relational grade. By using 277 grey relational analysis coupled with Taguchi approach optimal factor settings reported were 280 bTDC injection 278 timing, 180 bar injection pressure, 19 compression ratio and 80% of the full load and load was observed to be 279 most influential parameter. To validate the model developed for multi objective optimization confirmation test 280 were conducted and compared with prediction and the results were satisfactory. Further ANOVA was carried 281 out to explore relative influence of factors on responses and relative contribution of load was reported as 70.37%. 282 Thus the relationship between the diesel engine design and operating parameters with performance, combustion 283 and emission parameters could be better understood using Taguchi and grey relational method. 284

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