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1	Design of Piano-Playing Robotic Hand
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6 Abstract

Unlike the market slowdown of industrial robots, service entertainment robots have been 7 highly regarded by most robotics research and market research agencies. In this study we 8 developed a music playing robot (which can also work as a service robot) for public 9 performance. The research is mainly focused on the mechanical and electrical control of 10 piano-playing robot, the exploration of correlations among music theory, rhythm and piano 11 keys, and eventually the research on playing skill of keyboard instrument. The pianoplaying 12 robot is capable of control linear motor, servo-motor and pneumatic devices in accordance 13 with the notes and rhythm in order to drive the mechanical structure to proper positions for 14 pressing the keys and generating music. The devices used for this robot are mainly crucial 15 components produced by HIWIN Technology Corp. The design of robotic hand is based on 16 the direction of anthropomorphic hand such that five fingers will be used for playing piano. 17 The finger actuations include actions of finger rotation, finger pressing, and finger lifting; time 18 required for these 3 stages must meet the requirement of rhythm. The purpose of 19 entertainment robot can be achieved by playing electric piano with robotic hand, and we hope 20 this research can contribute to the development of domestic entertainment music playing 21 robots. 22

23

24 Index terms— entertainment robot, robotic hand, fast fingers, robotic palm, piano-playing robot.

25 1 Introduction

he trends of robotics research has been getting more popular in recent years, while there have been continuous appearance of various types of robots in domestic and international media with applications ranging from education, entertainment, medical care, to home care service. This is an indication that the scope of robot application has already been extended beyond the factory and entering our daily lives and various other fields.

The gradually aging population distribution in this century and decreasing annual birth rates among developing 30 countries have led to the aging society and dramatic change to the productivity-generating young/middle-31 aged/elderly population structures. Thus it can be predicted the imaginary era of robots in future world will 32 soon be developed and realized in human technology world following this trend. Unlike the market slowdown of 33 industrial robots, service & entertainment robots have been highly regarded by most robotics research and market 34 35 research agencies. In response to the trend and pulse of this era and based on the considerations of manufacturing 36 cost and market demand, education, leisure, and entertainment types of robots are going to be the top choices 37 for future investments. In this study we plan to focus our research on entertainment type keyboard instrument

38 playing robots.

Keyboard instrument includes harpsichord, organ, piano and electronic keyboard, while music tones of specific frequencies are generated by tapping keys by fingers. Keyboard instrument is suitable for solo or accompaniment with wide popularity among general public. Along with the enhanced living standards in our country, piano and electronic keyboard have become the important instruments for many families to encourage their children to learn how to play. In light of this popularity of keyboard instrument and the advancement of robotic technology, in this

project we plan to develop a robot which is capable of playing keyboard instrument. There have been many music 44 playing robots development in various countries. Toyota Motor Corporation (TMC) ??1] in Japan developed 45 Partner Robot in 2004 for the 2005 World Exposition in Aichi, Japan which was specialized in hand motions such 46 as music playing. The first generation of this robot is capable of playing trumpet for the purpose of edutainment. 47 The new model of Partner Robot [2] announced in December 2007 utilized technologies of precise control and 48 coordination for enhancing the flexibilities of palm and arm leading to additional violinplaying feature. The 49 Waseda University in Japan announced WABOT-2 [3] in 1984 as a robot capable of playing organ in accordance 50 with music standard. The next year ??ASUBOT [4] was announced with additional music score reading feature 51 and capability of playing keyboard instruments with 16 kinds of tunes. Although all aforementioned music robots 52 are capable of creating different music, they have been limited to mechanical finger motions or music playing 53 within small acoustic range, and the instruments in use are limited to certain mechanism while the entire acoustic 54 range has not been fully utilized. We hope the development of this keyboard instrument playing robot can be 55 different from others as a music robot which plays music faster without being limited by certain mechanism. We 56 would like to fully utilize the existing robotic design capability of our school in order to develop a product filled 57 with market potential. 58

59 We would like to have the electronic keyboard playing robot developed by this project to play the electronic 60 keyboard with anthropomorphic approach while utilizing linear motor, step motor and pneumatic cylinder to control the mechanism of music playing robot. The mechanical-electrical system we utilize is composed of 61 62 computer based controllers, where two sets of linear motors sharing the same driver board are used for drive two 63 palms. There are five fingers with each palm, and there are one joint mechanism and one finger mechanism in each finger controlled by step motor and pneumatic device. The step motor, pneumatic device and linear motor 64 are controlled by the system through FPGA controllers. Every note is generated with the robot touching the key 65 in four stages: palm movement (including finger rotation), finger pressing, finger holding, and finger releasing. 66 Upon receiving information of music note and rhythm, the system will determine next targeted position for music 67 playing based on music note information. Figure 1 is composed with anthropomorphic approach. 68

⁶⁹ 2 Design of Horizontal Hand Movement

In automatic production process the linear movement mechanism is the kind of most frequently seen application 70 carrier with degree of freedom of one to multiple dimensions. There is more than one kind of mechanism design 71 for selection. The classification based on movement style will lead to the categories of indirect transmission 72 and direct transmission. The indirect transmission is the most widely used form where rotating movement is 73 74 converted into linear movement through mechanical mechanism. The common examples of indirect transmission 75 are ball screw mechanism, rack and pinion mechanisms, and conveyor belt. The examples of direct transmission 76 include pneumatic device, hydraulic device and linear motor. For pneumatic and hydraulic devices to achieve energy transmission, fluid must be used as the media. This will not only increase the system complexity, but 77 78 also lead to various issues such as maintenance and regular service, media replacement, and increased cost. The linear motor with electromagnetic field as the media is capable of greatly reducing system complexity and issues 79 of maintenance and service. The industry has always been pursuing higher precision, more rapid production, and 80 less burden of maintenance, while linear motor can meet all these criteria. The contact driving system of linear 81 motor has introduced structural revolution for traditional tool machine, automation equipment and inspection 82 instruments with no need for bearings and couplings. 83

⁸⁴ 3 a) Analysis of hand moving time

There are numerous factors affecting the moving speed of the mechanical hand. First we can exclude the weight 85 factor and calculate the result of initial stage of hand speed. So far the hand promoter movements can follow 86 several factors such as the interval containing seven keys (C,D,E,F,G,A,B) obtained from research on fundamental 87 acoustic range, the time required for the hand to move from one interval to the next, the study on basic dimension 88 of piano, the length of piano keys, and the hand moving speed from the leftmost side to the right-most side. The 89 study shows there are two stages of the sequence of hand movement as shown in Figure 2: the speed of movement 90 from C5 to F5, and the speed of movement from F5 to C6 of the next interval. The movement of mechanical 91 hand is driven by the linear motor. A certain distance of mechanical hand movement will be set for calculation 92 of the speed, acceleration and time required for such movement. As one of the necessary factors for procurement 93 of linear motor, this setting will lead to more accurate speed of linear motor during the movement. 94

95 4 C5 D5 E5 F5 G5A5 B5 C6D6 E6

96

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This calculation is based on the relationship between speed and acceleration. The movement of linear motor is different from the operating approach of circular rotary motor such that the calculation of rotation speed and acceleration/deceleration is based on the principle of linear movement. During linear movement, the time needed for a stationary object to be accelerated to the maximum speed is called acceleration time. While the object is moving at maximum speed, it is called the speed of constant speed movement. The time needed for an object to decelerate from this constant speed to the stationary state is called deceleration time. The linear structure of linear motor will lead to fixed motion stroke such that there will be speed planning curves of acceleration, constant speed, and deceleration for the movement speed, and the acceleration, deceleration and time can be

constant speed, and decelerationcalculated by using the formula.

¹⁰⁶ 5 c) Calculation of specification selection

The following study is to derive required formula of acceleration a (2.1), formula of speed v (2.2), time t and speed curve formula (2.3) based on continuous thrust of linear motor, while the acceleration can be directly calculated from motor output and load. The motor output in this study is based on statistics of production specifications, while the load includes the custom-designed contained mass of mechanical hand, combination mass of motor promoter, and friction (which can be neglected in this study). The following analysis is focused on the time required for the LMC and LMS linear motors of HIWINMIKRO to move the mechanical by 16.5cm.

The LMC series of coreless motors are the first to be analyzed, where the calculation is based on the heaviest mass of contained mass of mechanical hand limited to 2kg. This numeric value is used as the calculation parameter.

¹¹⁶ 6 d) Result of specification selection

From the Table ?? below we find that the numeric values of each series of linear motor calculated based on the distance of 16.5cm (60 per minute and quarter note speed 0.1333s) can all meet the standard. But the difference between sizes of promoter volume is rather large. The same P60 quarter notes all meet standard time, but the volume or promoter of LMC series is smaller than that of LMS series. Thus LMC series is the more suitable linear motor for the piano playing robot of this study due to the visual, aesthetic and performance factors.

¹²² 7 Table 1 : Time of note

In this plan we use LMX1E-C linear motor platform with coreless motor which is suitable for high speed and 123 multi-axis simultaneous movement applications. The compact volume is the main feature, and the increment-124 125 type of analog or digital optical scale is used for the position feedback. LMX1E-C platform has superior dynamic features with length up to 4000 mm, maximum acceleration as high as 100m/s 2, and maximum speed as high 126 127 as 5m/s. Therefore it is the linear motor meeting the requirement of this project as shown in Figure 3. The application of common mechanical hand is limited to grabbing object with anthropomorphic approach. With 128 the space limit of hand design, many sensors and wirings have be installed in the hand such that the flexibility 129 of fingers are limited leading to the lack of dexterity when flexible finger actions are needed. In this study the 130 dexterity of finger has been demonstrated in the form of piano-playing. The piano-playing mechanical hand is 131 based on anthropomorphic design. The hand is slightly wider than human hand at 120 mm, while the appearance 132 of finger is imitating the bending of human finger during piano playing. On one hand it is an imitation of human 133 playing piano, on the other hand it allows only the finger pulp to touch the surface of piano key. The opening and 134 closing actions of hand during piano playing can demonstrate the technical aspect and anthropomorphic effect. 135 As for the chromatic keys, they can be reached by all fingers expect for the thumb, and this is the design which 136 has not been seen from previously announced piano-playing mechanical hands. 137

¹³⁸ 8 a) Mechanical hand system

The second generation mechanical hand system includes finger rotary actuator, finger-pressing pneumatic 139 cylinder, finger flexible connection, linear guideway, swinging arm protection device, bottom plate structure, and 140 sliding panel device. The structure of this mechanical hand is as shown in Figure 3.2 "The physical appearance 141 142 of the second generation hand", and detailed statistics refer to Table 3.1 "Table of performance parameters of mechanical hand". The bottom plate of the mechanical hand is mainly 6061-T6 aluminum plate with 4 sets of 143 pneumatic finger-pressing modules and an independent pneumatic thumbpressing module installed on top of it. 144 There are 4 SBR06 pneumatic cylinders beneath the hand bottom plate mainly for the extended actions of 4 sets 145 of modules. Thus there are linear guideways installed at the bottom of the 4 sets of pneumatic finger-pressing 146 modules mainly for allowing the 4 sets of pneumatic fingers to reach the chromatic keys. 147

The decomposition of finger pneumatic module of mechanical hand is based on single module, where the servo 148 is used as an actuator at the rotary joint. The actuator is installed in the rectangular groove on the sliding 149 bottom plate with threaded holes on both sides for fixing. When the signal of rotating angle is received by the 150 servo, it will drive the protection swinging arm on top of the servo. This protection swinging arm is installed at 151 the bottom of special rotary joint, and there are threads on the rotary joint for integration with auxiliary clamp 152 plate. There are bearing housings in the auxiliary clamp plate. This pressing plate will be placed on top of 153 154 rotary joint during installation. The auxiliary clamp plate is installed on the sliding plate where there are holes 155 for fixing and locking. Screws are utilized to penetrate through bearings until the rotary joint is screwed into the thread. This is mainly for reinforcement of the strength of rotary joint where the original structure of cantilever 156 beam is converted to shear free beam for distributing the force on both ends such that the prolonged stress will 157 not lead to the load of the servo motor. 158

There is the second degree of freedom on the rotary joint for the installation of finger rotation center. There are bearing housings in the fingers for installation of bearings. In the end the penetration of steel shaft is utilized

and the E-type ring is used for buckling mainly for avoiding axial movement. Inside the finger there are grooves 161 for integration with flexible device, where flexible devices are connected by using 1mm steel wire. The bouncing 162 back action of finger is achieved by placing compress springs around steel wire. During compression of springs 163 there must be guideways for guiding direction of spring compression such that the steel wire plays an important 164 role in the center of spring. As for the selection of spring, if the wire diameter is too large, the rigidity can be too 165 strong and causing fractures. If the spring is too soft, the finger bouncingback will not be fast enough to keep up 166 with the beat. The flexible device is connected to the combination of pneumatic cylinder and linear guideway. 167 The pneumatic cylinder is mainly for generating tension with respect to flexible device with stroke around 10mm. 168 The tension is controlled by flow adjustment by throttle valve on pneumatic cylinder. The direction of linear 169 guideway should be kept consistent with the auxiliary force, such that attention must be paid to the interference 170 issue during assembly of linear guideway and pneumatic cylinder, otherwise the response speed of fingers during 171 piano playing can be affected. 172

As for the identification of corresponding same music alphabet, the number can be used to classify the interval 173 level. The number "5" will be attached to the music alphabet of the key in the same set with central C. Numbers 174 such as "6", "7" are attached to the notes higher than the set of central C, while numbers such as "4", "3" are 175 attached to the notes lower than the set of central C as shown in Figure 4. For the finger rotation of piano-playing 176 177 mechanical hand, features such as fast rotation, precise position, and compact motor size are required, so is a 178 certain level or torque. Although the required torque force is not too large, certain level of support from rated 179 torque is required. Nonetheless, the most important factor is the size of motor. If the size of motor plus reducer box is too large, the designed mechanical hand will be too large. On the contrary, a smaller motor will lead to 180 a smaller mechanical hand which is more like a human hand. As for the reducer box, the finger motor does not 181 need a very large reduction ratio. 182

The design of second generation mechanical hand is different from the first generation where the torque 183 transmission by actuator is conducted by direction transmission approach instead of the connecting rod approach. 184 This is mainly due to the additional third degree of freedom installed on the movement joint within the mechanical 185 hand leading to the modified design of the second generation mechanical hand. The protection swinging arm is 186 installed on the servo axis such that there will not be any position sliding during finger rotation with gears of 187 swinging arm and servo axis meshing with each other. The screw fixation approach between protection swinging 188 arm and rotary joint is mainly for facilitating the replacement during maintenance. The principle of protection 189 swinging arm is when the finger is hit by external force, the mandatory rotation under the excitation state 190 of protection servo motor will lead to skipping of internal gear. The appearance dimensions of the actuator 191 installation are as shown in Figure 5. The finger rotation is transmitted to the rotary joint through protection 192 device. Since the protection device is a flexible connection part, reaction will be generated during finger pressing 193 process affecting the rotary joint. The protection device cannot withstand such force, thus the auxiliary clamp 194 plate is designed in response to this issue. The auxiliary clamp plate is installed on top of the rotary joint. The 195 reaction will cause the rotary joint to be lifted, and this is when the auxiliary clamp plate can balance the force. 196

¹⁹⁷ 9 c) Device design of mechanical hand-pressing actuator

For the finger pressing actions of piano-playing mechanical hand, features such as speed, stability, and compact 198 device size are required. By studying the action of finger pressing the piano key, we find that currently there 199 is no control over force for finger action such that the control device is only handling constant speed movement 200 with fixed stroke. In terms of the volume requirement, there must be 5 actuators placed on the hand for pulling 201 steel wires, thus lie-flat installation is required for the devices to be used in order to prevent the hand from being 202 too thick. In terms of the pressing speed, there must be very fast reciprocating motion in a rather short stroke 203 204 such that there will not be very high demand on precision. In terms of output force, this finger mechanism will require at least force of 1kgf and above. Summarizing features of all aforementioned device requirements, we 205 conclude that pneumatic cylinder is the best choice. It is equipped with aforementioned features of compact size, 206 short stroke, large force, and high speed. Pneumatic cylinder is a kind of pneumatic driver generating linear 207 movement which has been widely adopted in the industry. Because pneumatic cylinder is capable of generation 208 simple mechanical actions, there are various kinds of pneumatic cylinders with numerous methods of use. 209

In the design of finger pressing mechanism of mechanical hand, the pneumatic cylinder is used as the actuator 210 in coordination with other auxiliary devices. The reciprocating actions are generated by the pneumatic cylinder 211 to pull finger by pulling connection steel wires. With one end of the finger fixed on a rotation axis, the finger 212 motion of up and down swinging is centering on the rotation axis. When the steel wire is pulling the finger based 213 214 on the principle of leverage, it is a fixed ratio between the distance from the joint of steel wire and finger to the 215 rotation center, and the distance from finger tip to the rotation center. This is mainly the increase of pressing 216 speed which reduces the movement of pneumatic cylinder. Due to this ratio, the increased speed will lead to 217 reduced action force. When there is larger resistance at the finger tip, larger force is required at the stressing end for achieving balance due to the principle of force arm. For the overall beauty of fingers, this design also prevents 218 the exposed mechanism from being seen during actuation of mechanism such that the overall appearance is a 219 better fit for anthropomorphic design. This is going to contribute greatly to music playing where more lively 220 music can be played as the demonstration of finger dexterity while meeting the feature of anthropomorphic 221 fingers. 222

The difference between the design of the second generation mechanical finger and the first generation design 223 is at the position of bending joint. Based on the joint of human hand, the bending position of mechanical hand 224 is at the junction between carpal bone and mid bone. The distance from the finger bending joint to the finger 225 tip is 100mm, while the distance from the junction of steel wire and finger to the bending joint is 17mm. This 226 will lead to a larger ratio such that the pneumatic cylinder only needs to move by 5 mm for the pressing distance 227 to reach 25 mm. This way the speed of finger pressing on the piano key will be greatly enhanced to twice the 228 speed of the first generation. However, in terms of the pressing force, the adoption of old pneumatic cylinder 229 will lead to slightly insufficient force output. The reaction force is increased due to the enlarged force arm of 230 the principle of leverage such that greater force is required at the stressing end in order to achieve the balance. 231 Therefore, in the second generation design we replace the old pneumatic cylinder with the one with outer cylinder 232 diameter of 10mm in order to achieve ideal output force. Figure 6 is about the description of all parts of the 233 second generation finger, and the simulation of pressing action. New design approach has been used with respect 234 to the speed of piano key pressing by the second generation fingers. The distance from the finger bending joint 235 to the finger tip is 100mm, while the distance from the junction of steel wire and finger to the bending joint 236 is 17mm. The speed of finger pressing the piano key can be more obvious through the principle of leverage, 237 while the design of finger appearance has been also imitating the bending of human fingers while playing piano. 238 239 Through the computer-assisted mechanical drawing, 3D drawing has been utilized for the simulation of finger 240 pressing as shown in Figure ?? and Figure ??. Linear guideway has been utilized, while the coordination of 241 mechanism is based on the connection between pneumatic cylinder and linear guideway through steel wire clamp plate. The steel wire is fixed between the pneumatic cylinder and the linear guideway such that when the force is 242 transmitted from the pneumatic cylinder to the steel wire, it is not easy for the torque generated during pulling 243 of steel wire to lead to increased friction of the linear guideway. Based on this design, there is one module per 244 finger installed in the hand. Due to the size factor, the linear guideway must be fixed on the hanger in side-lying 245 fashion. There are 2 M2 threads on the slider for fixing the steel wire clamp plate on the slider and for fixing the 246 axis of pneumatic cylinder beneath the slider. There are two aluminum plates on the steel wire clamp plate for 247 clamping the steel wire. The double clamping approach can ensure the steel wire not to be loosened while being 248 pulled. The Figure 9 below is the description of pressing design of the second generation mechanical hand. In the 249 system design of the second generation mechanical hand, the opening angle is required by the fingers. Based on 250 the design principle of the first generation mechanical hand, the finger itself must be equipped with two degrees 251 of freedom. However, in the finger design of the second generation mechanical hand, a third degree of freedom 252 must be added, which is the position required for the fingers to hit the chromatic keys during piano playing, while 253 the appearance must be even more anthropomorphic. Based on these two criteria, the finger design of the second 254 generation mechanical hand must have 3 degrees of freedom, where the third degree of freedom is for the finger 255 to be extended to the position of chromatic key. The basic actuation principle is similar to the first generation 256 mechanical hand. The difference is the pneumatic cylinder is installed in the hand on a sliding module such that 257 the finger module can be moved as an entity. Therefore the pressing module itself must have two degrees of 258 freedom. In terms of the module, the action force generated by the pneumatic cylinder is transmitted through 259 the flexible device. The pneumatic cylinder is installed in the rear of the finger mainly for avoiding the friction 260 loss during force transmission. The flexible device we use is 1mm steel wire. The path of steel wire is almost 261 a straight line during transmission for generating enough force for finger pressing. When the finger is opened 262 by a certain angle, the steel wire will be bent with an angle. This angle is not very large, and the steel wire is 263 introduced into the rotary joint with the tangent approach, such that the transmission of steel wire is based on 264 an arc angle rather than a right-angle bending. This bending is within the bending range of 1mm steel wire. 265 The appearance of the second generation mechanical hand is as shown in Figure 10. Whenever the chromatic 266 keys need to be pressed during piano-playing, the mechanical hand will be moved to the proper position, while 267 the finger for pressing the chromatic key will be stretched to the position and the finger will be pressing down. 268 After the pressing action is completed, the hand will return to the normal state. The finger pressing process is 269 simulated by computer software as shown in the following Figure 11 and Figure 12. The distance from the finger 270 position on the white key to the finger position on the chromatic key is 30 mm, the width of chromatic key is 271 around 8 mm, and the width of buffer silicone at finger tip is 6 mm. Therefore, there is only 1 mm tolerance 272 of left/right deviation during finger pressing process, which is very important to the stability after stretched 273 mechanism. There is a drop height between the white key and the black key, which means the black key is 15 274 mm above the white key. Therefore, for fingers to reach for the chromatic keys while playing on white keys, they 275 must be lifted. However, the actuation of mechanism must be in a stable state such that the heights of fingers 276 and keys will be raised by a certain distance during pianoplaying on the white keys. The dimensions of the design 277 appearance are as shown in Figure 13 and Figure 14. In the mechanical hand design, fingers are designed with 278 modularization approach. A finger is designed to be a module capable of pressing the piano key and rotating by 279 a certain angle, such that a mechanical hand is composed of four identification finger modules and one thumb 280 module. As mentioned in previous chapters, the finger module includes servo motor, pneumatic cylinder, linear 281 guideway and flexible transmission device. All devices will be integrated on one mobile joint mechanism to form 282 a mobile joint module with the feature of the second generation mechanical hand as the third degree of freedom 283 for playing the chromatic key. The appearance dimension of this module is as shown in Figure 15. The bottom 284 plate of mechanical hand is integrated with the mobile joint module in order to facilitate the action of forward 285

stretching of mobile joint. Therefore there must be guideway at the bottom plate as the junction between the bottom plate and the mobile joint. Here we use the MGN5C guideway of HIWINMIKRO with 70 mm length. The action force is generated by the pneumatic cylinder during the movement. The pneumatic cylinder is designed to be installed at the bottom of mechanical hand where the mobile joint and the axis of pneumatic cylinder are integrated. The moving speed of this mobile joint is controlled by the exhaust throttle valve. The higher speed will lead to larger impact on the mechanical hand, thus this statistic must be adjusted in accordance with the requirement of piano-playing. The dimensions of the design appearance are as shown in Figure 16.

²⁹³ 10 Analysis on the Fingers of Mechanical Hand

The R&D of the fingers of mechanical hand has gone through three generations of designs such that the analysis on the fingers are focused on the comparisons among the structural strengths and stresses of these three generations, and the formula calculated from these stress related statistics. The analysis on the fingers through software approach and the results are as shown below.

²⁹⁸ 11 a) Static analysis of the fingers

According to the data of output file of ABAQUS/CAE, the maximum stress of mechanical finger takes place 299 at the element near the fixed end. Since in this study we focus all the loads of mechanical hand on one side, 300 there is going to be a rather significant phenomenon of stress concentration. The yield strength of aluminum 301 alloy material (6061-T6) is at 270MPa with must larger estimated safety factor mainly because of the better 302 processing properties of aluminum and easier access to such material. Thus there is less concern of the stress 303 damage to the mechanical fingers. With the smaller stress on the mechanical fingers, other materials (such as 304 PE material with rapid prototyping approach) with less strength can be used for saving the cost. However, there 305 is another consideration of using aluminum as the material of mechanical hand. During the stage of program 306 307 testing of mechanical fingers, if the fingers are not completely raised after pressing on the piano keys and the linear motor has started to drive the lateral movement of mechanical hand, the mechanical hand may suffer from 308 lateral impact due to large force generated by the linear motor. This is why we need a high safety factor for the 309 mechanical fingers. 310

The results of analysis on mechanical fingers are described in Table 2 and Table 3. The features of maximum stress and maximum deformation among three generations of mechanical fingers have led to different statistics of increased finger load. As for the stress curves and deformations of the second generation and third generation mechanical fingers, there is not much difference between them. However, there is obvious difference between their inner structures. There are weight reduction holes within the third generation fingers to greatly reduce the mass for finger pressing action without losing any strength.

Based on the experimental statistics of structures, stress of each load, and deformations of the first generation, 317 second generation, and third generation mechanical fingers, curves are drawn from all points for solving the 318 trend lines. Six linear straight lines can be obtained from Figure 17 Equations 5.1 to 5.3 are the formulas of 319 load and stress of mechanical hand structure, where y is the stress value based on structural analysis, and x is 320 the load (from 0.2kgf to 1kgf). This formula can be used for estimating the stress values of mechanical fingers 321 corresponding to all loads. Equations 5.4 and 5.6 are the formulas of load and deformation of mechanical finger 322 structure, where y is the deformation values based on structure analysis, and x is the load from 0.2kgf to 1kgf. 323 This formula can be used for estimating the value of deformation of mechanical finger structure corresponding 324 to all loads. 325

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 $^{^2 {\}rm J}$ e XV Issue I Version I



Figure 1: Figure 1 :

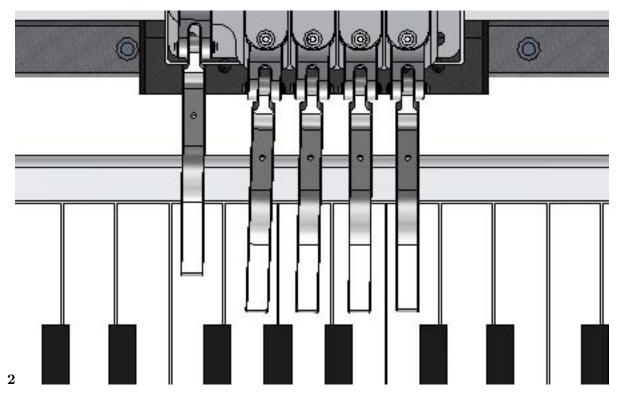


Figure 2: Figure 2 :



Figure 3: Figure 3 :

移動時間Ti	P30	P60	P90	P120
四分音符(1拍)	0.2666	0.1333	0. 0888	0.0666
八分音符(.5拍)	0.1333	0.0666	0.0444	0.0333
十六分音符(.25 拍)	0.0666	0.0333	0.0222	0.0166

Figure 4: Figure 4 :



 $\mathbf{5}$

Figure 5: Figure 5 :

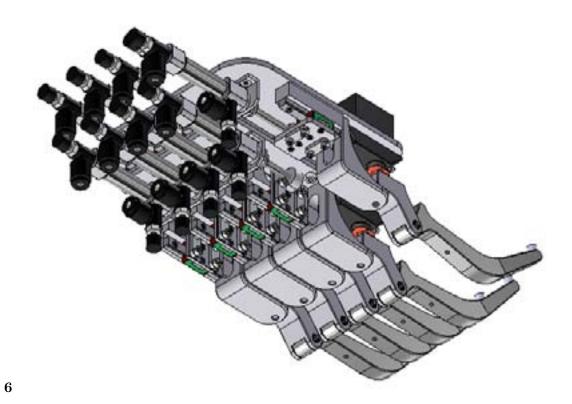
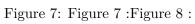
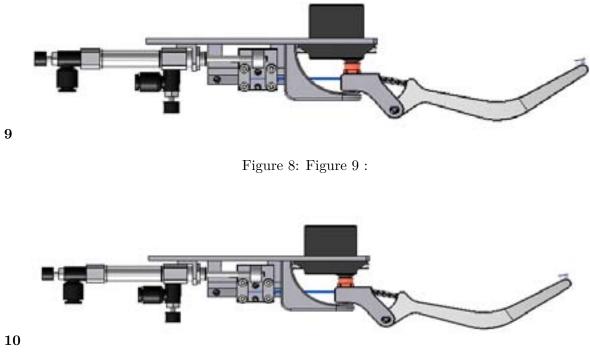


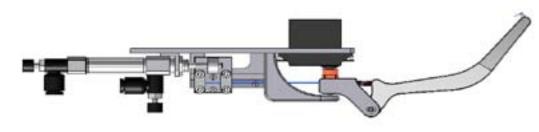
Figure 6: Figure 6 :



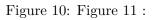












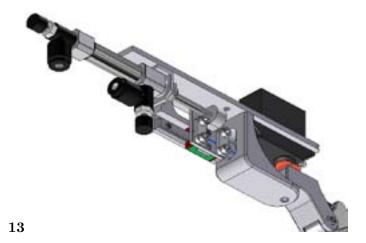


Figure 11: Figure 13 :

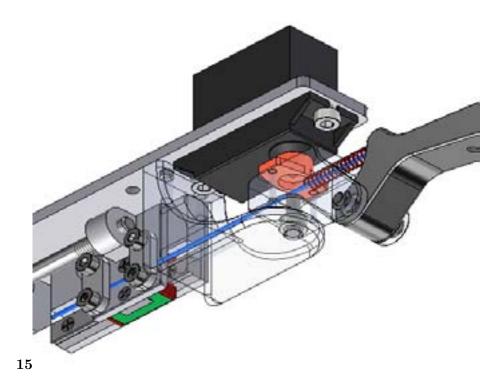


Figure 12: Figure 15 :

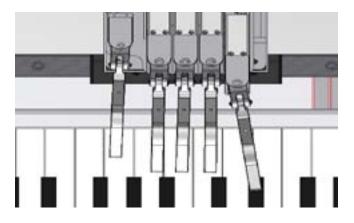


Figure 13:

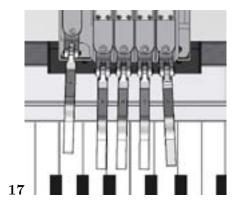


Figure 14: Figure 17 :

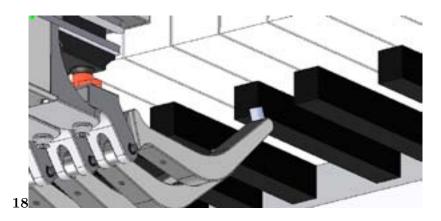


Figure 15: Figure 18 :

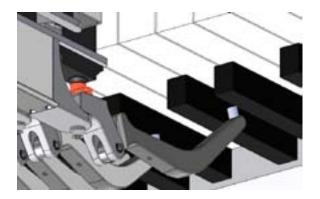
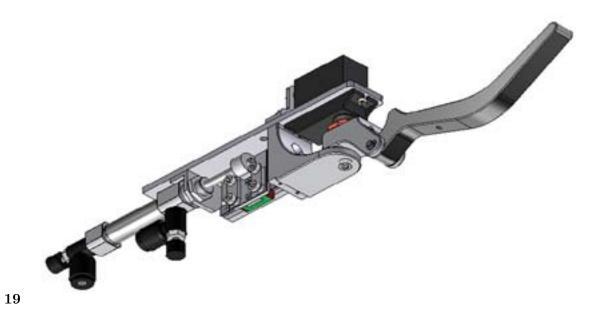
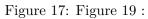


Figure 16:





$\mathbf{2}$

	mechanical finger (MPa)				
Load (kgf) Model	0.2	0.4	0.6	0.8	1
First					
generation	1.215	2.43	3.645	4.861	6.076
finger					
Second					
generation	$2.068 \ 4.136 \ 6.204$			8.273	10.34
finger					
Third					
generation	$2.122 \ 4.244 \ 6.367$			8.489	10.61
finger					

Figure 18: Table 2 :

3

Load (kgf) Model First generation finger	$0.2 \\ 0.0048$	$0.4 \\ 0.0097$	$\begin{array}{c} 0.6 \\ 0.0146 \end{array}$	$0.8 \\ 0.0194$	$\begin{array}{c}1\\0.02433\end{array}$
Second generation finger	0.0015	0.0308	0.0464	0.0617	0.0778
Third generation finger	0.0015	0.0315	0.0473	0.0631	0.0788

Figure 19: Table 3 :

326 .1 Acknowledgement

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