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Analysis of Thermal Resistance and Pumping Power of Rectangle Micro Channel Heat Sink for upper Flow with Different Coolant

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7 Abstract

In this paper we optimise the performance of microchannel heat sink with upper flow 8 arrangement of flow at entrance and exit. The performance of micro channel heat sink is 9 directly affected by the pumping power and the thermal resistance. Here we flow from the 10 upper section and optimise to be very low pumping power and thermal resistance. The aspect 11 ratio and the hydraulic diameter of the microchannel are same for flow arrangement. Fluid 12 flow and heat transfer are investigated on the basis of the simulation of the micro channel 13 with number of channel in rectangular shapes. The aim of this work is to get an impression of 14 the physical behaviour in small elements that enable the development of new liquid cooling 15 systems with higher cooling ability and higher effectiveness. 16

17

18 Index terms— upper flow arrangement, micro channel, very low pumping power and thermal resistance.

¹⁹ 1 Introduction

ince the pioneering work of Tuckerman and Pease [1] in 1981, many studies have been conducted on micro-channel 20 heat sinks as summarized by Phillips [2] and more recently, by Morini [3]. The need for cooling in high power 21 dissipation (100 W/cm 2) systems in several scientific and commercial applications such as microelectronics 22 requires something beyond the conventional cooling solutions. A number of studies have investigated the thermal 23 24 design optimization of micro-channel heat sinks to determine the geometric dimensions that give optimum 25 performance. For the heat transfer study purpose, the channel walls were assumed to behave as fins. With the increasing heat production of electronic devices, the air cooling technology reaches its limits, whereas liquid 26 cooling represents a promising opportunity to develop cooling devices with much higher heat transfer coefficient. 27 Today's rapid IT development requires high PC performance capable of processing more data and more speedily. 28 To meet this need, CPUs are assembled with more transistors, which are drawing more power and having much 29 higher clock rates. This leads to an ever-larger heat produced by the CPU in the computer, which will result 30 in a shortened life, malfunction and failure of CPU. The reliability of the electronic system will suffer if high 31 temperatures are permitted to exist. Therefore, removal of heat has become one of the most challenging issues 32 facing computer system designers today. However, conventional thermal management schemes such as air-cooling 33 with fans, liquid cooling [4], thermoelectric cooling [5][6][7][8][9], heat pipes [10], vapour chambers [11], and vapour 34 35 compression refrigeration [12] have either reached their practical application limit or are soon become impractical 36 for recently emerging electronic components. Therefore, exotic approaches were regarded as an alternative to 37 these conventional methods in sufficient for cooling further high power processors.

As the fluid is passing through the different section of the micro channel the distribution of the fluid in the passage is disturb the flow condition of the fluid that affect the velocity and thermal boundary layer of the flow. As flow is reached fully developed there is no change in the velocity of the fluid layer. The thermal and velocity boundary layer are playing an significant role in the fluid flow in micro channel. The different shapes of micro channel are used to dissipate the large amount of heat from the system or electronic circuit. As a practical cooling fluid, the liquid metal must satisfy the following requests: non-poisonous, non-caustic material, low

viscosity, high thermal conductivity and heat capacity. Most studies in this approach employed the classical fin 44 theory which models the solid walls separating microchannels as thin fins. The heat transfer process is simplified 45 as one-dimensional, constant convection heat transfer coefficient and uniform fluid temperature. However, the 46 nature of the heat transfer process in MCHS is conjugated heat conduction in the solid wall and convection to 47 the cooling fluid. Using a nano fluid as the heat transfer working fluid has gained much attention in recent 48 years. Xuan and Roetzel (2000) proposed two theoretical models to predict the heat transfer characteristics of 49 nano fluid flow in a tube. Li and Xuan (2002), Xuan and Li (2003) and Pak and Cho (1998) experimentally 50 measured the convection heat transfer and pressure drop for nano fluid tube flows. Their results indicated that 51 the heat transfer coefficient was greatly enhanced and depended on the flow Reynolds number, particle Peclet 52 number, particle size and shape, and particle volume fraction. These studies also indicated that the presence 53 of nano particles did not cause an extra pressure drop in the flow. Recently, Yang et al. (2005) carried out 54 an experimental study attempting to construct a heat transfer correlation among the parameters that affected 55 heat transfer. For a laminar flow regime in a circular tube, they indicated that the heat transfer effective for 56 the nano fluid flow had a lower increase than predicted by either the conventional heat transfer correlation for 57 the homogeneous or particle-suspended fluid. Ding et al. (??006) reported heat transfer effective data for 58 59 the force convection in circular tubes using carbontube (CNT) nano fluid. In most of the studies mentioned 60 above, the nano fluid heat transfer flow characteristics were carried out in macro-scale dimensions. Only a few 61 studies addressed the nano fluid flow and heat transfer in micro-scale dimensions. CHEIN AND HUNAG (2005) 62 EMPLOYED a macro-scale correlation to predict micro channel heat sink performance. In experimental aspect, Chein and Chuang (2007) studied the general behaviour heat sink performance and particle deposition effect 63 when nano fluid is used as the working fluid. In the study of lee and mudawar (2006), al2o3-h2o nanofluid was 64 used as working fluid. They pointed out that the high thermal conductivity of nano particles can enhance the 65 singlephase heat transfer coefficient, especially for the laminar flow. Due to complicated heat transfer phenomena 66 and large variety in nano fluids further studies on nano fluid flow and heat transfer characteristics in micro-scale 67 dimensions are still necessary. In this study, thermal resistance characterizing MCHS performance using nano 68 fluids as coolants are investigated. We particularly focus on the microchannel geometry effect on the MCHS 69 performance when nano fluid is used as the working fluid. Although micro -channel heat sinks are capable of 70 dissipating high heat fluxes, the small flow rate produces a large temperature rise along the flow direction in 71 both the solid and cooling fluid, which can be damaging to the temperature sensitive electronic components. 72 73 Therefore, more sophisticated predictions of the temperature field are essential for an effective micro-channel 74 heat sink design. A more accurate description of the heat transfer characteristics can only be obtained by direct numerical simulation of three dimensional fluid flow and heat transfer in both the solid and cooling fluid. 75

76 **2** II.

77 **3** Analysis Procedure

The micro Channel heat sink modelled in this investigation consists of three arrangement of fluid flow. The fluid is flow through the front, upper and the side of the channel there are two shape of micro channel heat sink are used. One is the rectangular shape and another is the trapezoidal shape are used. The aspect ratio and the hydraulic diameter for the rectangle and trapezoidal micro channel heat sink are assumed to be same. The arrangement of fluid flow is from the different sections are the front, upper and the side of the micro channel. The two different fluids are used one is the water and another is nano fluid with thermal conductivity 10 times of water. This investigation has to be carried out for the high performance of the micro channel.

These studies can help to clarify some of the variations in the previously published data and provide a fundamental insight into thermal and fluid transport process occurring in the microchannel heat sinks designed for electronic cooling and other application.

- 88 The analysis is based on the following assumptions:
- To simplify the analysis, the following assumptions are made in modelling the heat transfer in micro channels of the present study:
- 91 ? Steady state flow.
- 92 ? Incompressible fluid.
- 93 ? Laminar flow.
- 94 ? Constant properties of both fluids and solid.
- 95 ? Effects of viscous dissipation are negligible.

96 **4 III.**

97 5 Mathematical Formulation

The combination of the thermal resistance models and the optimisation algorithm served as useful tool in the design of the micro channel heat sink. The thermal resistance in the heat sink arises from three sources: conduction resistance in the heat sink, including the fin effects; convection resistance between the micro channel surfaces & the coolant & the resistance due to the temperature arise of the cooling fluid.

$_{102}$ 6 Rth= Rcond+Rconv+Rcap

103 (1)

The total thermal resistance is calculated:Rth=Tmax-Tmin/Q (2) ? = $v.\hat{I}$?"p(3) IV.

106 7 Computational Domain

107 A schematic of the rectangular micro channel heat sink is illustrated with upper flow arrangement.

108 8 Results and Discussion

109 9 Conclusion

This all analysis is done on the basis of simulation for the rectangular shape of micro channel heat sink for upper flow arrangement to investigate the role of thermal resistance and pumping power.

112 ? In this investigation concluded that comparisons between the water and custom nano fluids having 113 thermal conductivity 10 times more of water for rectangular shape of micro channel heat sink for upper flow 114 to predetermining the effect of pumping power and thermal resistance. ? Thermal resistance and pumping power 115 are the parameters that are depend upon the geometrical and flow parameters. ? In this investigation water 116 shows most predominant results as compare to the custom nano fluid. ? From this investigation we conclude 117 that there is very low value of thermal resistance and a low pumping power is required for the coolant used as 118 water.

119 VII.

120 10 Nomenclature

A Area exposed to heat transfer c p Specific heat (J kg? 1 K?⁻¹



Figure 1: Fig. 1:

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Figure 2: Fig. 2 :Fig. 3 :Fig. 4 :Fig. 5 :

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